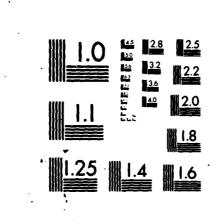
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REPORT NAEC-92-114

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REDUCTION OF EXHAUST SMOKE FROM GAS-TURBINE ENGINES
BY USING FUEL EMULSIONS

Propulsion Support Equipment Division Support Equipment Engineering Department Naval Air Engineering Center Lakehurst, New Jersey 08733

21 OCTOBER 1980

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REDUCTION OF EXHAUST SMOKE FROM GAS-TURBINE ENGINES BY USING FUEL ENGLISIONS

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20. ABSTRACT

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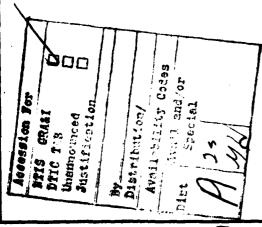
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The initial series of experiments suggested that a 48 percent reduction in exhaust particulate concentration, based upon a correlation due to Champagne, was possible with an emulsion having a water-to-fuel ratio of 0.1.

The later series of tests was oriented toward optimization of the water concentration for smoke reduction and examination of the complete engine power spectrum. Continuous decreases in exhaust smoke were observed up to the highest concentration tested, a 0.5 water-to-fuel ratio. The maximum reduction in exhaust particulate concentration was 80 percent based upon the Champagne correlation. Emulsions composed of 15 and 30 percent water-in-fuel ratio were tested throughout the engine power range, and smoke reductions were observed at all power points. The greatest reductions were found at the highest power points where the smoke problem is the greatest.

Combustion efficiency was calculated from the exhaust chemistry and decreased with the addition of water. The reduction in efficiency was very small at full power but became quite significant at the lower power conditions. Comprehensive measurements of gaseous exhaust emissions were also reported.

Within the limits of combustor rig testing, the water-in-fuel emulsion concept was shown to have a potential for significant reductions in exhaust smoke at the high power conditions where smoke is the greatest problem; the reductions in combustor performance were minimal at these conditions. The concept shows less potential at the lower power levels of operation. Since increasing the water concentration continued to reduce smoke, in actual engine operation the concentration could be tailored to meet the required smoke level.



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I. INTRODUCTION

Many present-day aircraft turbine engines produce exhaust with visible amounts of smoke. The objectionable conditions occur even though the smoke concentrations are very low and represent very small losses in combustion efficiency. Many recent studies have shown the directions necessary in combuster design for smoke abatement, and excellent results have been obtained. There remains, however, a problem in the testing of existing designs on stationary engine test stands where smoke production levels may be above environmental standards or desires.

Low-internal-phase-ratio emulsions of water-in-fuel have been investigated for their potential in reducing exhaust smoke from gas-turbine engines. The unique mechanism is the selective vaporization of the internal phase during the period of droplet heating; this vaporization and sudden expansion causes the fuel drop to break up into much smaller droplets. The potential for reducing soot formation in heterogeneous combustion is suggested by this increased atomization and dispersion of the fuel.

Experiments were conducted in a combustor facility fabricated from T-63 engine hardware using a single-can combustor with a dual-orifice pressure atomizer. Two series of tests were performed. The initial sequence employed air flow conditions that simulated full engine power (the smokiest condition. The second series employed air flow characteristics at reduced power conditions. During the experiments, the fuel specifications, water concentration, surfactant concentration, and dispersion size were varied. Moreover, measurements of exhaust smoke, combustor temperature rise, flame radiation, and combustion efficiency based upon exhaust chemistry were obtained.

II. SUMMARY

A. During the experiments, it was observed that a concentration of 2% surfactant was sufficient to produce a stable emulsion that allowed significant smoke reduction; no benefit could be ascribed to the use of larger quantities. No effect of dispersion size on smoke reduction was directly observed, although the homogenizing equipment allowed only a small range of dispersion size to be investigated. Fuels which were characterized by high aromatic content and high boiling range were studied; the smoke reduction concept was found to be equally effective in both cases. Specifically, the initial series of experiments suggested that a 48% reduction in exhaust particulate concentration, based upon a correlation due to Champagne, was possible with an emulsion having a water-to-fuel ratio of 0.1. The flame radiation was reduced by 20%, and the reductions in combustor temperature rise and combustion efficiency were minimal. Since there was evidence of a continuing decrease in exhaust smoke with further increases in water concentration, an additional group of experiments was conducted.

- B. The later series of tests were oriented toward optimization of the water concentration for smoke reduction and examination of the complete engine power spectrum. Continuous decreases in exhaust smoke were observed up to the highest concentration tested, a 0.5 water-to-fuel ratio. The maximum reduction in exhaust particulate concentration was 80% based upon the Champagne correlation. Emulsions composed of 15% and 30% water-in-fuel ratios were tested throughout the engine power range, and smoke reductions were observed at all power points. The greatest reductions were found at the highest power points where the smoke problem is the greatest.
- C. Comprehensive measurements of gaseous exhaust emissions were also obtained. NO_X was reduced by increases in water concentration with the greatest reductions occurring at the higher power levels. CO and unburned hydrocarbons increased with water concentration; the increases were small at full power but became increasingly large at the lower power conditions. Combustion efficiency was calculated from the exhaust chemistry and decreased with the addition of water. The reduction in efficiency was very small at full power but became quite significant at the lower power conditions.
- D. Within the limits of combustor rig testing, the water-in-fuel emulsion concept was shown to have a potential for significant reductions in exhaust smoke at the high power conditions where smoke is the greatest problem; the reductions in combustor performance were minimal at these conditions. The concept shows less potential at the lower power levels of operation, but smoke is not usually a problem during low power operation. Since increasing the water concentration continued to reduce smoke, in actual engine operation the concentration could be tailored to meet the required smoke level. Moreover, it is recommended that full-scale engine tests be conducted for the purpose of understanding the effects that water-fuel emulsions have on engine horsepower/operation and exhaust plume visibility.

III. DISCUSSION AND THEORY

A. SMOKE PRODUCTION IN TURBINE ENGINE COMBUSTORS

1. The smoke produced by a gas turbine engine is an aerosol of soot or carbon particles resulting from the incomplete combustion of the fuel. Objectional conditions occur even though the particulate concentrations are very low, typically less than 0.005% by weight, and represent only very small losses in combustion efficiency (1).* The major contribution is formed in the primary zone, but soot may be generated in any part of the combustor where mixing is inadequate and fuel-rich pockets exist. Essentially, the carbon loses to the more active and more available hydrogen in the competition for available oxygen. Most of this fine carbon or soot is consumed in the secondary and quench zones where there is an abundance of air; the remainder becomes exhaust smoke.

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^{*}Numbers in parenthesis indicate items within the Bibliography.

- 2. The physical characteristics of the fuel spray have an important effect on the production of soot. Larger droplets have a longer lifetime, causing higher heat absorption and enhancing soot formation. Reduced spray penetration and cone angle serve to increase the fuel/air ratio in the region of the nozzle leading to an increase in soot production (3).
- 3. The hydrocarbon structure of the fuel is also known to have a significant effect on the production of soot. The higher molecular weight compounds are more prone to producing soot because of their higher carbon-hydrogen ratio; also they are less volatile and are often pyrolyzed before distilling out of the droplet. The existence of smaller droplets would allow the heavy ends to vaporize sooner so improved atomization may very well reduce this source. The aromatic compounds have a tendency to produce smoke about 30 times greater than that of the paraffins for the same boiling range (7) due to the relatively high stability of the carbon ring (16). This is an important consideration because of the latitude for aromatic composition in JP-5, 0-25% by volume.

B. SMOKE REDUCTION TECHNIQUES

- 1. Several techniques are generally used to reduce the smoke from new combustor designs: increased mixing and a leaner primary zone to reduce the production of soot and increased residence times in the secondary and quench zones to promote complete burning of the soot (2,3). Considerable success has also been obtained by using the so called "air blast" atomizer, which not only provides a smaller droplet size, but provides a source of oxygen immediately in the region of the nozzle orifice thus reducing the fuel-rich pocket found with pressure atomizers (3).
- 2. These solutions do not solve the problem associated with the testing of existing designs on stationary engine test stands where smoke production levels may exceed environmental standards. There are several fuel additives available which act to suppress smoke, such as Ethyl Corporation's Combustor Improver #2 and ferrocene. The former is a manganese additive and has potential toxicity problems as well as leading to the accumulation of manganese oxides on critical areas of the turbine; ferrocene also tends to create deposits, but they are sometimes acceptable (4, 5, 6).

C. "MICRO-EXPLOSION" PHENOMENA

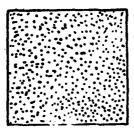
1. The combustion properties of drops of fuel emulsions were first investigated experimentally by Ivanov and Nefedov (14). Burning drops of water-in-oil emulsions and using high-speed cinematography, they showed that the more volatile water vaporized inside the drop as it was heated. The expansion of the water vapor violently tore the drop apart; this "micro-explosion" scattered very small droplets from the parent drop, increasing the total burning rate and reducing the carbon residue. Dryer (22) has

recently reproduced this experiment under a grant from the National Science Foundation.

2. The potential for reducing soot formation from gas-turbine engines is suggested by this increase in atomization and dispersion of fuel. Dryer (22) reports several investigations of using fuel emulsions for the reduction of soot from furnaces and boilers; these all involved the use of fuel oils that are heavier than kerosene type jet fuels (JP-5). To the author's knowledge there has been no previous use of emulsions for the reduction of smoke from aircraft turbine engines.

D. CHARACTERISTICS OF EMULSIFIED FUELS

 The type of emulsion used in this program is known as a lowinternal-phase-ratio emulsion of water-in-oil. That is the "dispersed phase", water, is a relatively small fraction of the system and the "continuous phase", fuel, makes up the bulk of the system. The illustration below compares the structure of such an emulsion with its opposite, a highinternal-phase-ratio emulsion. Chemicals known as surfactants are usually required to stabilize emulsions. They are typically a type of molecule which is soluble in water on one end and soluble in oil on the other end. A proper balance of this Hydrophilic-Lipophilic property must be attained to achieve a stable emulsion; an HLB number is assigned to each surfactant to characterize it. The implications of this system are beyond the scope of this work. ICI American has published a description of the system (20). For reference, an HLB of 5.3 was found to be satisfactory. For lowinternal-phase-ratio emulsions, the surfactant is usally ionic, so that the apparent charge can help prevent agglomeration and combustion which speeds separation.



Low-internal-phase-ratio water-in-oil



High-internal-phase-ratio oil-in-water

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2. Low-internal-phase-ratio emulsions are desirable for the type of work discussed here for two reasons: the change in viscosity is not great and the characteristic size of the dispersed phase (water droplets) is smaller. The reason for the first is obvious; it was found that emulsifying 10% water in the JP-5 raised the viscosity from 1.6 c $^{\rm S}$ to 2.0 c $^{\rm S}$. (Highinternal-phase-ratio emulsions usually have very high viscosities and non-Newtonian flow properties.) The second is inherent to the "micro-explosion" phenomena: the size of the dispersed phase must be much smaller than the mean diameter of the fuel spray so that spray drops will contain the emulsion. Emulsions of 10% water in JP-5, stabilized with a surfactant, were examined under a microscope, and the dispersed water droplets were found to be in the range of 1/2 to 2 microns. Since the SMD of the fuel spray was about 85 microns, the spray remains an emulsion. Figure 1 shows two photo micrographs of emulsions created under different conditions. A circle is superimposed to give an indication of the interior of an 85 micron fuel drop. If the dispersion size becomes the same order as a spray drop, the free surfaces of the emulsion will interfere with normal droplet formation process, and quite probably, the drops will not be emulsions, but either pure fuel or water.

IV. EQUIPMENT AND PROCEDURES

A. EXPERIMENTAL EQUIPMENT. With the exception of the fuel emulsification system, all of the experimental equipment used in this program was existing equipment already being used in turbine-fuels research at the U.S. Army Fuels and Lubricants Research Laboratory (AFLRL).

1. FUEL EMULSIFICATION SYSTEM

- a. The system used to make the emulsions was a Model 100 Laboratory Homogenizer manufactured by the Gaulin Corporation. It was chosen over other methods, such as ultrasonics, because it provided a capability for varying the dispersion size of the emulsion. Basically, the unit is a high-pressure, positive displacement pump which discharges a crudely mixed medium through a special homogenizing valve (see Figure 1). The mixture is accelerated through the orifice to strike the impact ring at velocities up to 300 m/sec (950 ft/sec); this action shatters the mixture into a dispersion of very small droplets. This dispersion size can be varied by adjusting the orifice size and pressure drop; examples of the resulting emulsions are illustrated in Figure 2.
- b. The homogenizer unit was integrated into the fuel system as an in-line emulsification system, thereby reducing the emulsion stability requirements. The system is shown schematically in Figure 3. The fuel and water are crudely mixed in the "mixing tee" upstream of the homogenizer. The first bypass system recycles excess flow because the pump operates at a constant flow rate of 4.1 l/min (65 gph) whereas the combustor only requires 1.9 l/min (30 gph) at the operating conditions used. The second bypass is

used to establish the correct flow rates of the two liquids before introducing the fuel into the combustor. The accumulator is necessary to dampen the pulsations in line pressure caused by the piston-compressor action of the homogenizer.

- 2. EMULSION ANALYSIS. The emulsions were analyzed for dispersion size by using a microscope with 430x magnification. A micrometer slide with 0.1 mm divisions was used to calibrate the scale on the eyepiece. Each division on the eyepiece was equivalent to about .003 mm so droplets of .001 mm could be measured to an accuracy of .0005 mm.
- 3. COMBUSTOR RIG. The combustor rig used for this study is based on engine hardware from the Allison T-63 engine used in the Navy's TH-57A helicopter. The burner is a single-can type with a dual orifice pressure atomizer centered in the dome as shown in Figure 4. At the exit of the burner can there is a centerbody which diverts the flow into an annulus where the nozzles and turbine blades are normally located. Gas-sampling probes, pressure probes, and thermocouples were arranged circumferentially in one plane of this annulus at various radial positions, as shown in Figure 5. Figure 6 shows the combustor rig as installed and instrumented in the AFLRL combustor lab. Table 1 presents the air flow and fuel flow conditions that were established to correspond with various power points following the guidelines of the manufacturer.
- 4. FLAME RADIATION MEASUREMENT. Total flame radiation was measured with Model R-2065 Asymptotic Radiometer manufactured by Hy-Cal Engineering. Figure 7 illustrates the installation of this unit. The window is sapphire to permit response to IR radiation out to around 5 microns; this is necessary to see the $\rm CO_2$ radiation (19). The important feature is that the window is flush with the combustor liner wall and has a 1500 viewing angle. Thus, it sees essentially the entire flame zone and measures the total radiation heat load to the wall at that point. This wide viewing angle is important in programs where the temperature patterns may shift due to changes in air-fuel mixing.
- 5. DATA ACQUISITION SYSTEM. The heart of the data acquisition system is a Hewlett-Packard 9820 programmable calculator; this is coupled to a scanner and digital voltmeter to automatically acquire data and process it. Operating conditions are then printed out for monitoring on a thermal line printer with an update about every ten seconds. The flow rates of the combustor air and fuel were measured with turbine flowmeters. All pressures were sensed with strain gauge pressure transducers activated by regulated power supplies. Ch-omel-alumel thermocouples, referenced to a 150°F regulated oven, were used for temperature measurement.
- 6. EXHAUST ANALYSIS INSTRUMENTATION. Exhaust emissions were measured on-line using the instruments listed on the following page.

Sample	Instrument	Sensitivity
Carbon Monoxide	Beckman Model 315B NDIR	50 ppm to 16%
Carbon Dioxide	Beckman Model 315B NDIR	300 ppm to 16%
Unburned Hydrocarbons	Beckman Model 402 FID Hydrocarbon Analyzer	0.5 ppm to 10% (CH ₄)
Nitric Oxide	Thermo-Electron 10A Che- miluminescence Analyzer	3 ppm to 10,000 ppm
Total Oxides of Nitrogen	Thermo-Electron 10A Che- miluminescence Analyzer with NO _X Convertor	3 ppm to 10,000 ppm
0xygen	Beckman Fieldlab Oxygen Analyzer	0.1 ppm to 100%

The exhaust sample was brought to the instruments through a 350° F heated teflon line and then appropriately distributed.

- 7. SMOKE ANALYSIS SYSTEM. The system used for measuring exhaust smoke level was designed according to the requirements of SAE-ARP1179. Briefly, a sample of the exhaust is passed through a strip of filter paper. Particulates from the exhaust are trapped on the surface, leaving a spot ranging in "grayness" from white to black, depending on the sample size and particulate content of the exhaust. The spot is then evaluated with a reflectometer. Refer to VB3 for detailed calculation.
- 8. <u>FUELS</u>. Three fuels of the JP-5 type were used in this program. Two of the three test fuels were specially blended at a local refinery to accentuate the two "smoke sources" within the fuel as previously discussed, i.e., one fuel had a high boiling range and a low aromatic content while the other had a high aromatic content but a lower boiling range. The third test fuel was simply a production-run JP-5 from Ashland Refinery. Table 2 compares the properties of these fuels to the JP-5 specification.
- B. <u>PROCEDURES</u>. The objective of this program was to determine the potential of reducing the exhaust smoke from a gas-turbine combustion chamber by emulsifying the fuel with water. The program was separated into seven major phases:
 - Phase 1 -- Formation and Characterization of Emulsions
 - Phase 2 -- Combustor Testing to Evaluate Potential for Smoke Reduction

- Phase 3 -- Sensitivity of Concept to Pertinent Fuel Properties
- Phase 4 -- Support Data on Best Candidate Emulsion
- Phase 5 -- Effect of Best Emulsified Fuel on Combustion Throughout the Power Spectrum
- Phase 6 -- Evaluation of the Effects of Particulate Emission Rates on Exhaust Plume Visibility
- Phase 7 -- Estimate of Effects During Full-Scale Engine Tests
- l. In Phase 1, the formation and characterization of stable emulsions of water-in-fuel were investigated. Possible candidate emulsion systems were identified, and an in-line system was developed to create emulsions of variable concentration and quality.
- 2. During the second phase, these emulsions were tested in a combustor rig operated at the smokiest condition to determine their potential in reducing exhaust smoke and to assess the effects of the fuel-modification on combustor performance, i.e., flame radiation, exhaust emissions, combustion efficiency, and temperature rise. The fuel used in the second phase was a "production run" JP-5.
- 3. In Phase 3, two specially blended JP-5 fuels were used to determine if the concept was sensitive to boiling-range end-point or aromatic content. These variables constitute two major fuel-related sources of soot formation in combustors.
- 4. Phase 4 was devoted to further combustor tests using additional water concentrations. Both 100% and 25% power levels were examined in an attempt to identify the emulsion characteristics that were optimum for smoke reduction.
- 5. During Phase 5, the combustor was operated over the entire power spectrum using the base fuel and fuel emulsions considered appropriate for full-scale engine testing.
- 6. The effort during Phase 6 was devoted to estimation of probable plume visibility on the basis of measured particulate emission rates.
- 7. All of the information acquired during the program was utilized during Phase 7. In this activity, estimates were made of the probable effect of fuel-water emulsions on plume visibility and particulate emission rates during full-scale engine tests.

The work associated with Phases 1-3 was performed during 1975, and the results have been reported previously on an interim basis. This report encompasses the entire effort, Phases 1-7, and includes all information previously submitted.

V. ANALYSIS

- A. PRESENTATION OF DATA. There are two types of data presentation: the test reports of the individual combustor experiments and the smoke data. Since the smoke data is derived from analysis and curve fitting of smoke spots on filter paper, it is not included in the individual test report which is immediate output from the calculator at the completion of each test.
- 1. The test reports are presented as Figures 8 through 62 and give summaries of the combustor operating conditions, a survey of the exhaust thermocouple measurements, the exhaust chemistry and the combustion efficiencies. Average values and standard deviations of the air and fuel flow parameters are compared with the desired engine parameters as given in Table 1.
- 2. The least-squares curve fits of the smoke spot readings for all the experiments are presented in Figures 63 through 77. Many of the experiments are combined on the various figures to help illustrate the effects on smoke reduction.

B. COMBUSTION PARAMETERS

1. Combustion Efficiency. Combustion efficiencies are calculated from the exhaust gas analysis according to a relationship developed by Hardin (11):

$$n_b = \left[1 - \frac{A \cdot f(UBH) - 121,745 \cdot f(CO - 38,880 \cdot f(NO) - 14,654 \cdot f(NO_2)}{A \cdot [f(CO_2) + f(CO) + f(UBH)]}\right] \cdot 100\%$$

where f(i) is the concentration of "i", in the exhaust and A is a constant based on the heat of combustion and hydrogen/carbon ratio of the fuel.

2. Flow Rate. The pressure and flow rate cannot always be attained exactly; in such cases the air flow loading factor is the critical scaling parameter which is matched along with the air/fuel ratio and the inlet temperature. The flow parameter is defined as:

$$\mathbf{FF} \equiv \frac{\mathbf{\omega} \sqrt{\mathbf{T}}}{\mathbf{P}}$$

where W = air flow rate

The same of the sa

T = temperature

P = air pressure

It is a measure of the mach number of the inlet air flow and hence the residence time in the combustor. This is a standard scaling method used by engine manufacturers (10).

3. Smoke Level. The sample particulate matter, as aforementioned in IVA7, is evaluated using a reflectometer. The calculation which assigns a smoke number (SN) to the sample is as follows:

$$SN = 100 \left(1 - \frac{R_{\rm g}}{R_{\rm w}}\right)$$

where $R_{\rm g}$ and $R_{\rm w}$ are the diffuse reflectance of the sample spot and the clean filter paper. Exhaust samples are taken over a range of sample sizes around W/A = 0.023 pound of sample per square inch of filter area. The resulting smoke numbers are plotted against log (W/A). These are least-squares fitted with a straight line; the interpolated value of SN at W/A = 0.023 is the reported smoke number for the engine operation condition. Champagne (9) gives a complete description of the procedure and relates the results to particulate concentration and exhaust plume visibility. Troth et al (10) provide a numerical relationship for that correlation:

 $d_8 = a_1 \exp (a_2 SN) [1-\exp (-a_3 SN)] + a_4 \exp [-a_5 (SN-a_6)^2]$

where $d_g = true smoke density, mg/m^3$

SN = EPA Smoke Number

 $a_1 = 0.8$

 $a_2 = 0.057565$

 $a_3 = 0.1335$

 $a_4 = 0.0942$

 $a_5 = 0.005$

 $a_6 = 27.5$

VI. DISCUSSION OF EXPERIMENTAL RESULTS

A. COMBUSTOR OPERATING CONDITIONS

- 1. Examination of the test reports shows that the air and fuel flow parameters were all quite stable during the tests which normally take about 20 minutes to complete because of the lengthy smoke measurements. (Unfortunately, due to an error in the programming, the "air flow loading factor" was printed out as "0.00" in the first seven experiments. Calculated values are written in.) The differences in fuel flow rate and air/fuel ratio account for the water and surfactant added to the fuel. The actual fuel flow rate was kept the same for all experiments conducted at similar power points. Another deviation in the printed output from the calculator was encountered during the second series of experiments. The method of reporting hydrocarbon emissions was inconsistent with the computer program; corrections have been made on the experiment test reports.
- 2. Except for the noted variances, a comparison of the operating conditions shows very little variation among the experiments. They are therefore felt to be a valid set of experiments upon which to base conclusions about the use of fuel emulsions to reduce exhaust smoke.

B. CHARACTERIZATION OF EMULSIONS

- 1. Four emulsion characteristics were found to be important in this program:
 - a. surfactant type,
 - b. surfactant concentration,
 - c. water concentration, and
 - d. dispersion size.
- 2. The first has already been discussed briefly. An HLB kit was purchased and used to find the desirable value for a water-in-JP5 emulsion. A surfactant with an HLB of 5.3 (90% SPAN 80*/10% TWEEN 80*) was used for the combustion experiments.
- 3. The surfactant concentration was also found to have an effect on the stability. ("Stable" here means the emulsion exhibits no separation to the unaided eye; agglomeration and coescence on a microscopic scale always occur to some extent.) The amount of surfactant necessary depended on the concentration of water. Emulsions of 10% water-in-JP5 could be stabilized for up to 12 hours with 2% of the above surfactant, whereas emulsions of 20% and 30% water showed separation in about 15 minutes. Only concentrations of 5% and 10% water were used in the early combustion experiments. Surfactant levels were maintained at the 2% value during the later tests, which included water-to-fuel ratios up to 50%. The in-line homogenizer unit allowed stable emulsions to be maintained during the performance of the tests.

^{*} Trademarks of ICI America, Inc.

- 4. It should be noted that the percentage definitions are related to the fuel flow above; the total flow of liquid to the combustor was not used as the reference. Thus, the term "20% emulsion" implies a mixture of one part of water to five parts of fuel on a volume basis. Surfactant concentrations are also based upon a measured volume of fuel, and the surfactant was mixed with the fuel prior to initiation of the combustion tests.
- 5. The dispersion size could be varied by changing the pressure drop across the homogenizer valve as previously discussed. The two photomicrographs of Figure 2 show the dispersions for the pressure drops of 2600 and 200 psi—the two extremes used in the combustion experiments. The scale is 3 microns per division for both pictures. The high-pressure case indicates dispersions of around 1 to 2 microns, whereas the dispersion in the other case is 5 to 10 microns with a few larger sizes apparent.
- C. <u>COMBUSTION EXPERIMENTS</u>. In addition to the individual test reports for each experiment, the salient features of the experimental program are summarized in Tables 3 and 4. The following discussion treats several pertinent aspects of the use of emulsified fuels in turbine combustors.

1. Exhaust Smoke

- a. Without exception, the addition of water to the fuel in the form of an emulsion resulted in a reduction in exhaust smoke. The results from the early series of experiments, summarized in Figure 78, suggest that significant reduction in smoke can be achieved through the addition of modest quantities of water. Furthermore, the trend of these results implies that larger quantities of water yield further smoke reduction. The more recent experiments confirm this tendency; results are shown for water/fuel ratios up to 50% in Figures 79 and 80. Throughout this series of tests, smoke was further reduced for each increase in water concentration. Although the shape of the curve implies that a limiting value exists, higher concentrations were not attempted since the physical properties of emulsions begin to change at concentrations not too far above this (18).
- b. Since the test results indicate that exhaust smoke decreases monotonically as the water concentration increases, there is no clearly defined optimum value for the water content of the emulsion. In practice, the selection of an appropriate blend would be governed by the magnitude of the smoke emission problem and by practical considerations associated with the engine fuel supply system. A body of data was acquired which describes the degree of smoke reduction available over the entire engine operating range. Water-in-fuel ratios of 0.15 and 0.30 were utilized; the results are described in Figures 81 and 82. The full power point corresponds to maximum smoke, and it is particularly significant to note that the greatest smoke reductions were achieved at the high power points. The lower power points are characterized by smaller smoke reductions, but the smoke levels are also low under low power conditions. In terms of smoke number, at the full power point, reduction by a factor of approximately 2 is possible at the 0.15 water-in-fuel ratio, while the addition of 0.30 water-in-fuel ratio allows smoke reduction by a factor of about 3.

- c. During the early series of experiments, an attempt was made to produce emulsions having different dispersion sizes by varying the homogenizer pressure drop. Estimated dispersion sizes ranged from 1 to 10 microns; these values are much smaller than the SMD of the spray. When these mixtures were tested, no effect of dispersion size was observed. However, it is probable that the relatively unstable emulsions produced with low surfactant concentrations are characterized by larger dispersion sizes. The results shown in Figure 78 indicate that the smoke reduction is affected by surfactant concentrations below a level of about 2%. Thus, it may be inferred that there is some effect of dispersion size, but the effect is quite small if the surfactant concentration and initial dispersion are sufficient to create a stable emulsion.
- 2. Combustor Temperature Rise. There is no apparent effect on temperature rise; this could be expected considering the very small amount of water actually added. With an overall fuel/air ratio of 0.0198, an addition of 12.1% wt (10% vol) of water based on the fuel is only an addition of 0.24% wt of the total flow. Assuming the specific heat of water vapor is twice that of air, the resultant decrease in temperature rise should be about 0.5%. If the temperature rise is typically about 630° C (1170°F), the effect is only 3° C (6° F)!
- 3. Combustion Efficiency. Combustion efficiencies were calculated from the combustor exhaust chemistry, and the results are shown in Figures 83 and 84. At full power, the efficiency is reduced by less than 1% at water/fuel ratios of 0.50. However, the effect of water addition is more noticeable for part load operation. As shown in Figure 84, the combustion efficiency for a 0.30 water-in-fuel emulsion at the 10% power point is about 4% lower than the value associated with the base fuel.
- 4. Flame Radiation. Burning emulsions of 10% water consistently resulted in about a 20% reduction in flame radiation. This is consistent with the idea that less soot is being produced in the primary zone.
- 5. Exhaust Chemistry. Measurements of exhaust concentrations were obtained for unburned hydrocarbons, carbon monoxide, and oxides of nitrogen. The results are summarized, with respect to water fuel ratio and power point, in Figures 85 through 90. In general, it may be observed that an increase in the water concentration corresponds to an increase in emission of unburned hydrocarbon and carbon monoxide and a decrease in emission of oxides of nitrogen. The changes are significant; the emission of oxides of nitrogen can be halved by the addition of 40% water, but the cost of this reduction is an increase of the same magnitude in emission of hydrocarbons. The observed trend corresponds to the effect of a cooler flame zone. The cooler flame may be due either to a water quench effect or to changes in the mixing of air and fuel. The increase dispersion of the fuel as a result of micro-explosions would reduce the level of high temperature diffusion-zone combustion and promote cooler premixed combustion.

D. <u>SENSITIVITY TO FUEL PROPERTIES</u>. The reductions in smoke brought about by emulsifying the fuel were equally effective with high aromatic fuels and fuels with high end points as evidenced by experiments 16 through 19. Therefore, no problems are foreseen in the application of this concept due to variations in fuel properties.

VII. EVALUATION OF CONCEPT AND APPLICATION OF RESULTS

Two correlations, attributed to Champagne (9) and Kelly (21), have been located to establish the correspondence between smoke number and plume visibility. Champagne provides an indication of the size of a visible plume in terms of smoke number (or smoke concentration). The correlation due to Kelly is a summary of data obtained from Navy jet engine test facilities. Both of the correlations are presented in Figure 91. Obviously, reductions in smoke number coincide with reductions in plume visibility, but the actual relationship depends upon the engine and installation. This program has shown that fuel emulsions can be used to reduce exhaust smoke with negligible effects on combustor performance. Therefore, it is concluded that there would be reductions in plume visibility if such a fuel is used in a full-scale engine, but the extent of the reduction cannot be predicted.

The concept of water-in-fuel emulsions reducing exhaust smoke from gas turbine engines appears to have its greatest application at full power conditions where the smoke problem is most severe. The greatest reductions in smoke were obtained under full power conditions, and the effect of water addition on combustion efficiency and combustor temperature rise was smallest at these points. Because the smoke levels were found to be monotonically decreasing with increased water concentration, it appears that the addition of water can be tailored to meet the level of smoke being produced.

The adaption of the technique to full-scale engine testing could be accomplished in two ways: if the fuel control system is flexible enough to accommodate the required increase in fuel flow, the easiest method would be to emulsify the fuel upstream of the high-pressure pump; the other possibility is to emulsify the fuel between the fuel control and the nozzle ring by temporarily replacing a section of the high-pressure line with an emulsification system similar to the one used in this program. The choice may depend on the individual engine.

VIII. RECOMMENDATION

Two important performance items cannot be quantified on a combustor test facility; these factors are (1) the effects on engine horsepower and operation, and (2) the effects on exhaust plume visibility. It is therefore recommended that full-scale engine tests be conducted in order to define these features of water-fuel emulsion time.

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		TABLE 1 -	T-63 COMBUS	TABLE 1 - T-63 COMBUSTOR RIG OPERATING CONDITIONS	CONDITIONS			
		BIP	BIT	Wa (17/6)	ωξ γα/m (1b/m)	F/A	ñ.	BOT C (F)
Mode	& Power	kpa (psia)		Kg/s (10/s)	/P / 1 / 1 / 1 / 1 / 1 / 1 / 1 / 1 / 1 /			
Ground Idle	10	252 (36.6) 149 (300)	149 (300)	0.699 (1.54)	0.46 (1.01)	.0109	1.158	561 (1042)
;	25	311 (45.1)	178 (353)	0.0821 (1.81)	0.59 (1.31)	.0121	1.145	648 (1199)
	40	363 (52.6)	204 (400)	0.948 (2.09)	1.74 (1.64)	.0131	1.163	(1290)
	S S	417 (60.5)	221 (430)	1.03 (2.27)	0.89 (1.97)	.0145	1.147	759 (1399)
Ciuise Ciiah/Hover	75	461 (66.9)	244 (472)	1.12 (2.46)	1.11 (2.45)	,0166	1.123	848 (1559)
Takeoff	2 00	525 (76.2)	273 (524)	1.21 (2.66)	1.43 (3.16)	.0198	1.094	971 (1780)

Nomenclature

Burner inlet air pressure Burner inlet air temperature Air flow rate

Fuel flow rate Fuel/air ratio wa volt/BIP Typical burner outlet temperature

TABLE 2 - FUEL PROPERTIES

	_	FUE	L*	
	JP5-HBR	JP5-HA	JPS-P	JPS-Spec.
Composition				
Saturate, normal	18.8	15.8	5.8	
Saturate, iso- and cyclo-	67.8	59.9	80.9	
Aromatic	13.4	24.3	13.3	<25%
Volatility				
Distillation				
Initial (°F)	390	366	356	
10%	406	380	384	400 max
20%	412	384	394	
50%	430	394	422	
90%	488	436	476	
95%	510	466	490	
Final	536	496 [*]	504	550 max
Flash Point, °F	162	145	142	140 min
Gravity, API (60°F)	42.7	42.1	42.4	36 to 48
Specific Gravity (75°F)	.81	.81	.81	.788 to .845
Fluidity				
Freeze Point, °F	-22	-52.6	-54	-51 max
Viscosity (100°F)	1.78	1.37	1.59	
Combustion				
Aniline Gravity Product Heat of Combustion, BTU/lb	6700 19,827	5759 19,702	6285 19,7 5 7	4500 min 18,300 min

^{*}JP5-HA was the high aromatic JP5, JP5-HBR was the high boiling range JP5, and JP5-P was the production JP5.

TABLE 3 - SUMMARY OF COMBUSTOR EXPERIMENTS

PROGRAM PHASES 1-3 All Runs at Full Power Conditions

Exper. No.	Dete	Base Fuel	Mater Conc. Wolf	Surfactant Conc. Wolf	APS rost	S &	Sel	는 당 기	831	Smoke No.	Particulates Conc. 'Amg/m	Combustion Efficiency	Combustor Temp. Rise	Combustor Flame Radiation
-	6/13/75	195-9	a	0	;	57.5	61.5	2,9	0.033	24.4	3.22	99.6195	661 (1221)	:
~	6/13/75	4-547	9	~	2600	49.0	53.0	7.4	0.036	16.9	1.95	99.5618	638 (1181)	:
•	6/13/75	JP5-P	2	~	1600	45.0	55.5	8.9	0.040	17.0	1.96	99.5254	637 (1179)	:
•	6/13/75	JPS-P	9	7	200	43.5	\$4.5	6.0	0.040	17.3	2.01	99.5311	632 (1170)	:
ď	\$7,8179	195.P	G	đ	;	\$2.5	58.0	3.6	0.032	25.5	3.45	99.6121	623 (1153)	9.61
•	6/18/75	195-P	2		2600	4.5	51.5	8.	0.038	18.5	2.19	99.5484	637 (1178)	15.9
	6/18/75	JPS-P	2	1/2	2600	7 :0	56.2	7	0.033	21.5	2.68	9609.66	. 645 (1193)	16.7
•	6/19/75	JPS-P	•		;	\$2.6	58.5	3.4	0.033	27.1	3.80	99.6000	618 (1144)	:
•	6/23/75	3.54C	G	G	:	58.0	63.5	2.2	0.032	27.6	3.93	99.6344	647 (1197)	19.1
5	6/23/75	JPS-P	• •	~	:	51.5	58.3	3.E	0.035	9.92	3.69	99.5690	632 (1175)	19.1
=	\$175/15	185.0	G	a	;	55.4	61.5	4.2	0.032	27.2	3.62	99.6142	634 (1174)	18.6
: 2	6/25/75	JPS-P	2	• •••	2600	45.0	52.5	5.7	0.040	18.9	2.25	99.5225	633 (1172)	14.2
1	6/25/75	JPS-P	·•	~	2600	51.7	59.5	N. A.	0.033	21.4	2.66	99.6193	646 (1195)	15.4
=	6/25/75	JPS-P	w		2600	50. S	5715	3.6	0.035	23.0	2.95	99.5672	633 (1171-)	14.7
2	6/25/75	JPS-P	s	. 2/1	2600	52.5	58.5	3.6	0.038	24.0	3.14	99.5838	628 (1163)	14.5
7	6/27/75	JPS-HA	6	•	:	60.5	8.5	2.5	0.033	33.6	5.55	99.6019	632 (1175)	17.9
2	6/27/75	JP5-14	2	~	2600	41.5	54.5	S . u	0.039	23.8	3.11	99.5286	623 (1170)	1.1
=	6/27/75	JPS-188	•	•	:	52.5	60.0	2.5	0.036	28.0	3.35	99.5741	(0511) 129 .	14.6
2	6/27/75	JPS-HDR	9	~	2600	45.0	51.6	4.9	0.041	15.4	1.74	99.5011	622 (1152)	8.11
		1												

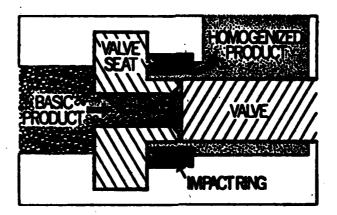
 JPS-P was the production JPS, JPS-MA was the high arcmetic JPS, and JPS-HRR was the high boiling range JPS.

2. Calculated from Champagae's correlation to Smoke Number (Ref. 9).

Pressure drop across "Hampenizer velva" which affects dispersion size.

TABLE 4 - SUMMARY OF COMBUSTOR EXPERIMENTS
PROGRAM PHASES 4-7
Fuel For All Runs Includes Surfactant 2% By Volume
Homogenizer Valve Pressure Drop = 2600 PSI For All Emulsions

Base Fue!
0 100 55.5 5 100 46.5
100
20 100
30 100
0 100
~
0 75
15 75
30
0 55
30
0
15 40
30 40
0 25
15 25
30 25
0
15 10
30 10

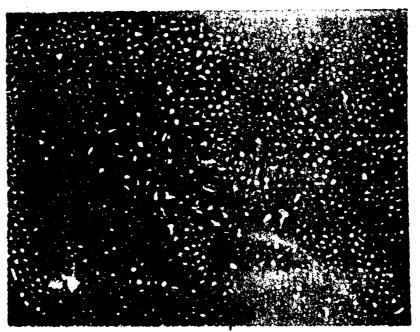


Close-up of a Gaulin homogenizing valve section. Product enters the valve area at high pressure. The pressure forces open the pre-loaded adjustable valve and the product passes through the aperture where an instantaneous pressure drop to less than an atmosphere occurs, causing shearing action and cavitation bubbles. The product then strikes the impact ring at a velocity of about 57,000 ft/min, further shattering the particles by impact and implosion of the bubbles. The homogenized product is discharged at a pressure sufficient for movement to the next processing stage.

(from Bibliog 4)

Figure 1 - Mechanism of Emulsion Formation





Photomicrographs of emulsions formed with two different homogenizing pressures. (1) Upper: $\Delta P = 2600$ ps., (2) lower: 200 psi. Scale: 3 microns/division. Circle is scaled to an 85-micron diameter to illustrate a spray drop of emulsion.

Figure 2 - Photomicrographs of Emulsions Showing Variation in Dispersion Size

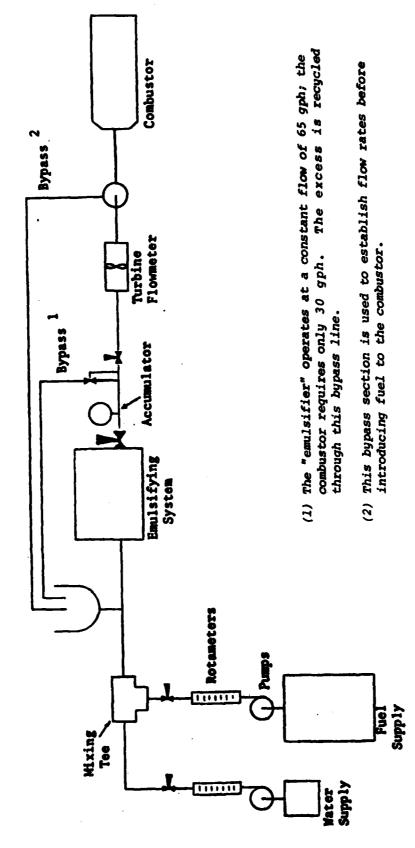


Figure 3 - Flow Diagram of Experimental In-Line Fuel Emulsification System



Figure 4 - T-63 Combustor Liner



Figure 5 - Exhaust Instrumentation Section

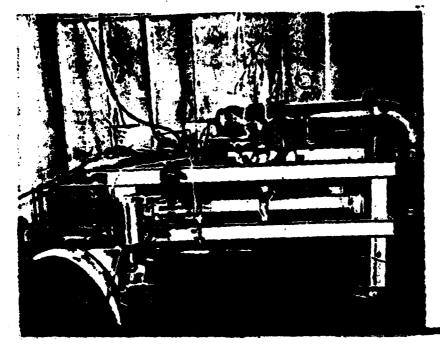


Figure 6 T-63 Combustor Rig Installation

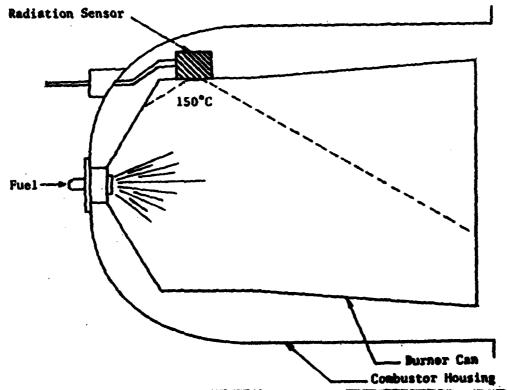


Figure 7 - Flame Radiation Measurement

THE STATE OF THE EMUSIONS FOR THE PEDICTOR OF THE PRINCIPLE OF THE PEDICTOR OF THE PRINCIPLE OF THE PRINCIPL	### FEET FUEL: 95-P FAULTHINGS COMBUSTOR STSTEM: 7-63 COMBUSTOR STSTEM: 17-63 COMBUSTOR STSTEM: 1-63 ***** EXPERIMENTAL TEST COMBITIONS ****** INLET AIR FREESURE, PSIG 71-29 78-29 FUEL FLOW RATE, LBS-NEC 77-29 78-20 FUEL FLOW LOADING FACTOR 77-29 78-20 FUEL PRESSURE**** FUEL PRESSURE*** FUEL PRESSURE** FUEL PRESSURE* FUEL PRESSURE** FUEL PRESSURE* FUEL PRE	* BURNER OUTLET TEMPERATURE SURVEY *	### ##################################
* 9. S. MAYY FUELS & LUBRICANTS RESEARCH LABORATORY TUBBINE CAMBUSTOR FACILITY STUDY OF FUEL EMBLSIONS FOR THE REDUCTION OF EXHRUST PARTICULATES *** U.S. WAYY BIR EMILINEERING CENTER	### 1013 73 **********************************	* BUTHER OUTLET TEMPERATURE SURVEY *	\ \\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\

LHSOPATOR?	
6. MENT FUELS & LUBRICARIA SUSEMENT YUNGINE CONSUSTATE FACILITY	֡
FUELS & L	
S. BPIII	
. : '	

THEN AN THE ENVESTORS FOR THE REDUCTION OF ENABLY PARTICLANTES

;

TIME: 11:31 POWER POINT: 1002 10% WATER DATE: 6/13/75 COTOLSTOR SISTEMS 7-63 TEST FUEL: JPS-P EMULSIFIED HITH

LAST SCHN 71.50 324.11 2.473 3.317 6.0000 26.20 324.00 3.160 3.160 1.000 1.000 1.000 1.000 FEST CONDITIONS: ****

71.89 .37
523.92 1.32
5.474 .012
3.327 .028
9.0224 .0022 ***** EXPERIMENTAL INLET MIR PRESSURE, PSIA INLET MIR TEMFERATURE, DEG F MIR FLOM RATE, LBS/SEC FUEL PLOM RATE, LBS/MIM FUEL MIR RATIO AIR FLOM LOADING FACTOR

FUEL PRESSURE: G. 440 FUEL TEMPERATURE=123.3 DEG F

- BURNER OUTLET TEMPERATURE SURVEY -

BOT MYC=1703.4 BEG F BOT HOT SPOT: TC# S = 1837 BEG BOT PATTERN FACTOR = .1137 **2 ANNEX.US DUTER PHANLUS** THER PHANUS CENTER 31

02.14.6 % UBH... 6.8 PPH \FRDPAHE +02.13.5 PFM <HOX-NO> Ni 99.5254 .021554 COMMUSTION EFFICIENCY, CALCULATED FROM EXHRUST CHEMISTRY: .02. PERIMES: Honoloniter processes C02..4.50 % NOX.. 55.5 PPH /6.9 3000 15.

* EXMANST CHEMISTRY *

Figure 10 - Test Report of Experiment No.

1. S. APMY FUELS C. LUSSICARITS RESIDANT P. LABUTATOP. TURBINE CONSUSTOF PACILITY.

JAN. ERULSIONS FOR THE REDUCTION OF EXHBUST PREFICULATES. U.S. MAYY AIR ENGINEETING LENTER. 3 Srub;

DRIE: 8-13 75 COMEUSTOR SYSTEM: T-63 POWER POINT: 1004 TEST FUEL: JPS-P ENULSIFIED WITH 104 WATER

FEST COMBITIONS ***
72.28 .43
522.79 .122
2.467 .038
.02219 .00022
0.2219 .00022 . 43 1. 22 . 013 . 08622 0. 00622 INLET MIR PRESSURE, PSIA
INLET AIR TEMPERATURE, DEG F
AIR FLUM RATE, LBS/SEC
FUEL FLUM RATE, LBS/MIN
FUEL AIR RATIO
AIP FLUM LONDING FACTOR

**** EXPERIMENTAL

LAST SCAN 72.88 523.68 2.443 3.293 9.2247

FUEL PRESSURE - NOT. FUEL TEMPERATURE - 97.4 DEG F

. BURNER OUTLET TEMPERATURE SURVEY .

1808 DEC .0976 Average . BOT ATC=1693.4 DEC F BOT HOT SPOT: TC# 5 BOT PRITERN FACTOR = CENTER ANNULUS **OUTER ANNULUS** THINE AMOLUS

02..14.8 % UBH.. 6.8 PPR < PROPRIE: NO2..11.0 PPR < NOX-NO> * EXHAUST. CHEMISTRY * CO2..4.50 % NOX.. 54.5 PPH 56. . 649 . CO2.

Homogeniger Aressee 5° 8.

Figure 11 - Test Report of Experiment No.

FUELS & LUBRICARITS FASSACH LABORATORY TUPBING CONBUSTOF FACILITY CO. THE REDUCTION OF SCHOOLS PARTICULATES - 15. NAVY AIR ENGINESPINAL CENTER	TIME: 10:18 POWER PO:NT: 100% ED HITH 10% WATER	MENTAL TEST CONDITIONS FY.38 .42 76.20 71.18 71.38 .42 76.20 71.18 71.39 .42 76.70 71.39 .43 76.70 71.39 .43 76.70 71.39 .43 76.70 71.39 .912 76.70 71.39 .92 76.70 71.39 .43 76.70 71.39 .912 76.70 71.39 .92 .93 71.39 .43 71.30	1838 DEG F -1134 -1734 -1734 -1735 -	EXMANST CHEMISTRY * 42 % 02.15.1 % UBM. 5.3 PPM <propake. %="" (nox-no)="" 51.5="" 7.4="" 92.0451<="" 98.5464="" 99.5464="" calculated="" chemistry:="" ed="" exhaust="" from="" mg2.="" ppm="" th=""></propake.>
U. S. MRHY.	ENTE: 6:18.75 COLEUSION STREMT T-63 TEST FUEL: UPS-P ENULSIFIED	***** EXPERIMENTAL ALLET RIR PRESSURE, PSIA INLET RIR TEMPERATURE, DEC F MIN FLON RATE, LBS.SEC MIN FLON RATE, LBS.YEC MIN FLON RATE, LBS.YIN MIN FLON LOADING FACTOR FUEL PRESSURE* * BURNER OUTL * BURNER OUTL	BOT AVG=1704.5 DEG F BOT HOT SPOT: TCB S = 18 BOT PATTER! FACTOR = .11 OUTER ANNULUS 1 13 CENTER ANNULUS 2 11 1NNER ANNULUS 3 11 11 11 11	CO
C. S. ARMY FUELS & LUGITCANTS RECEARING LIBOPATOR: TURBINE COMBUSTOR FACILITY - 37UD ' V. UEL ENULSIONS FOR THE REDUCTION OF EXHAUST PHYSICULATES U.S. MAVY AIR ENGINEERING CENTER	UNENTER 6/19.75 CONENTRY S'S (SH: 7-63 PONEP FOINT: 1002 TEST FUEL: JPS-P SECTION STATEMENT STAT		BOT RYCE-1653.4 DEG F BOT PRITERN FACTOR = .1541 TOB NVERACE STD DEV TOB NVERACE STD DEC TOB NVERACE STD DEV TOB NVERACE STD D	• EXMAUST CHENISTRY • CO032 % CO24.33 % OZ15.2 % UBM 3.6 PPH :PROPRIE INO52.7 PPH NOX 59.0 PPH NOZ. 5.5 PFH (NOX-NO) SHOKE INTRÉE! 25.5 COMPANYION EFFICIENCY, CALCULATED FROM EXHAUST CHENISTRY: 99.6121 ". REMARKS: REMARKS:

Figure 13 - Test Report of Experiment No. 6

Figure 12 - Test Report of Experiment No. 5

TORBINE COABUSTOR FROM LACENTORS TORBINE COABUSTOR FROM LACENTORS TORBINE COABUSTOR FROM LACENTORS O.S. MAYT MIR ENGINEERING CLATER CHEBUSTOR STOFF T-63 PONER PCINTS 1062	#### EXPERIMENTAL TEST CONDITIONS +*** INLET AIR FRESSUPE, PSIA 71.22 .27 76.20 71.10 AIR FLOW RATE, LBS/SEC 2.495 .014 2.660 2.477 FUEL FLOW RATE, LBS/HIM 2.995 .014 2.660 2.477 FUEL FLOW LOADING FACTOR 1.0996 .0064 1.0940 1.0930 FUEL PRESSURE= 0 FUEL PRESSURE= 0 FUEL PRESSURE= 87.9 DEG F	* BURILEP OUTLET TEMPERATURE SURVEY * 60T AVG=1645, 2 DEG F 80T HOT SPUT: TCB 8 = 1796 DEG F 80T HOT SPUT: TCB 8 = 1736 OUTEP ALMULUS 1 13 1700 13 1418 CENTER ANNULUS 2 1772 13 1418 CENTER ANNULUS 2 1772 14 1581 11 1581	* EXHAUST CHEMISTRY * 1.1. 52.6 PF:1 NOX 53.5 PPH NOZ 5.9 PF:1 CHON-NO. TRANSPORT CHISTON EXHAUST CHEMISTRY: 3.4 FPH CPROPAGE. S.10 PF:1 CHON-NO. TRANSPORT CHEMISTRY: 0.9526 Figure 15 - Test Report of Experiment No. 8
The fuels to the recent of the control of the contr	INLET NIR PRESSURE, PSIA PSIA		UST CHE PPH 02 ULATED ROW EXSE

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LHECZATOPY
LUBALLANTS PERLIPORT COMERSTOR FACTLETS
S. HRHY FIELS & TURBINE

*** COURT OF CALL EMBESSIONS NOT THE CENTER OF THE TRANSPORT PRATICULATED HAVE THE U.S. MAYOR FOR DESIGNED CENTER.

DATE: 6/25.75 CONSUSTOP STSTER: T-63 TEST FUEL: JPE-P

TIME: 10:58 POWER FOINT: 100%

FUSL: JPE-P **** EXPERIMENTAL (EST CONDITIONS

INLET AIR PRESSURE, PSIA 72.83 76.20 76.20 76.20 10.00 FELLIANT SCHOOL FOR THE PRESSURE, PSIA 72.83 76.20 76

FUEL PRESSURE" 0 FUEL TEMPERATURE 85.7 DEG F . BURNER OUTLET TEMPERATURE SURVEY

DEG F	GE STB DEV 7 17 7 42	411 411 411			9 9 9 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
5 F 1828		7 1633 13 :	5 179	11 162 14 176 3	9 12 163 178 178
BOT AVG=1654.2 DE C BOT HOT SPOT: TC	OUTER ANNUALUS	•	CENTER PHANLUS	Deize Annus	· ·:

* EXHAUST CHEMISTRY *

CO... 1832 % CO2..4.50 % O2..14.7 % UBH.. 2.2 PPN <PROFME>NO.. 58.0 PPN HOX.. 63.5 PPN NO2.. 5.5 PPN <NOX-NO>SPINE NYBER: 27.6

COMBLESTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 99,6344 % FUEL HIR ROTIO.CALCULATED FROM EXHAUST CHEMISTRY: ... 021477 REMORES:

Figure 16 - Test Report of Experiment No. 9

1 S. ARIN FUELS & LUBPICANTS FELENAPH LABORATORS TO STATE TORBINE CONSUSTOR PACIFIC

*** SOUNDER DEL EROLSTONS FOR THE REDUCTION OF CHAINST PARTICULATES U.S. HAVY HIR ENGINEERING CENTER

DPTE: 6.23/79 CUIGGSTOR SYSTER: T-63 TEST FUEL: UPS-P

TIME: 11:36 POWER POINT: :00%

	72.50 524.33	2.502	1.0628
•	76.20 26.20	2.669	1.0940
(Tinks ++++	.62	956	.00039
	72.90 525.72	2.529	1.1028
**** EXPERIMENTAL	ET AIR PRESSURE, PSIA ET AIR TENPERATURE, DEG F	E FLOW RATE, LBS/SEC	FUEL-AIR RATIO AIR FLOW LOADING FACTOR

FUEL PRESSURE* 0 FUEL TEMPERATURE*125.8 DEG F * BURNER OUTLET TEMPERATURE SURVEY *

	STD DEV	_	173	21	8 2-	53	. 51		9:	196	205	. 96	33
# 1784 DEG F	ų		Φ 6	1623	1731	1750	1784	265	92/1	9 63	0	8	1221
BOT HOT SPOT: TCB 8 -	בייון בייון	CUTER ANNULUS 1	•	- 92	13 . CENTER PANNULUS 2	n	₩,	end (◆1	· AINTER WHITE S	•	21	C

. EXHAUST CHEMISTRY +

Figure 17 - Test Report of Experiment No. 10

5. S. APMY FUELS & LUGSTERATS ELEGACH LIBORATORY TURETHE CALESTIVE TACTLITY

COLL ENULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATED U.S. NAVY ATM ENGINEERING CENTER ŝ 9

TINE: 9: 8 POWER POINT: 1002 10.5% 6.35.75 Greatstop States: 1-63 Test Fuel: Joseph

72.40 524.11 2.530 2.981 .01964 26.20 224.03 224.03 3.160 1.0960 STD. DE: **** EXPERIMENTAL INLE: AIR PRESSURE, PSIA LILET AIR TEMPERATURE, DEG F AIR FLOM RATE, LBS/SEC FUEL FLOW RATE, LBS/MIN FUEL AIR RATE)

FUEL PRESSURE* 0 FUEL TEMPERATURE* 86.1 DEG F

* BURNER OUTLET TEMPERATURE SURVEY *

BCT A:G=1662.1 DEG F
BOT HUT SPOT: 1CB 8 = 1815 DEG F
801 PATTERN FACTOR = 1343 :
10178 ANEMALUS 1 1778
1 1778 CENTER. MIRILUS IMER FANKLUS 35

02.15.2 % UBH. 3.4 PPH <PROPAHE: NO2. 6.1 PPH <NOX-NO> UNTRUSTION EFFICIENCY, CALCULATED FROM EXMANST CHEMISTRY: 99.6142 FULLAIR RATIO, CALCULATED FROM EXMANST CHEMISTRY: .020502 FEMENSS: * EXHPUST CHEMISTRY * CO2..4.35 % WOX.. 61.5 PPH

Figure 18 - Test Report of Experiment No. 11

÷ FORTH FUELS IN LIGHT STATES ASSESSED ANGOMINED. ; .

OF THE EMPISIONS FOR THE REMOVED OF A MASS PARTICULATER USES OF A MASS PARTICULATER.

LAST SCAN 72.50 325.62 2.588 3.377 .02244 76.26 76.26 524.66 2.666 3.338 3.338 Unit: 5:25.7 CALCISTOR SYSTAN: 1-63 TEST FUEL: JPS-P SHULSTFIED WITH 10% WATER FEST COMPITIONS ### RVERACE STD.DEV. 72.36 .28 524.85 .11.18 2.316 .012 3.224 .012 3.224 .012 1.0679 .0075 **** EXPERIMENTAL INLET AIR PPESSUES, PSIA INLET AIR TEMPERATURE, DEG F AIR FLOW RNTE, LBS/SEC FUEL FLOW RNTS, LBS/NIN FUEL/AIR RATIO AIR FLOW LORDING FACTOR

* BURNER OUTLET TEMPERATURE SURVEY ..

FUEL PRESSURE* 0 FUEL TEMPERATURE*118.9 DEG F

1 * 1820 DEG F 1 1590 AVEPACE ST BOT AVG-1642.6 DEG F BOT HOT SPOT: TC# 8 BO: PRITERN FACTUR = CENTER RIMMLUS OUTER ANNULUS INIER ANNULUL

CO2.4.48 % 02.14.4 % USH. 5.7 PFN < PROPAME: NOX. 52.5 PPH NO2. 7.5 PPN < NOX-NO. * EXHAUST CHEMISTRY ..

CL SUSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 99.5225 FULL HIR RATIO, CALCULATED FROM EXHAUST CHEMISTRY: .021490 3000 P.S.1.

Figure 19 - Test Report of Experiment No. 12

NAEC-92-114

EUSITEART STATES CONTROL L'EXPATORS CUMENSTAR FAILLIT. THE REDUCTION OF LAMEST PARTICULATES * 1	TIME: 11146 POWER POINT: 100% IX WATER	CONDITIONS ***** BESIRED LAST SCAN ************************************	•	TEMPERATURE SURVEY .	•			•	.5 % UBH. 3.6 PPH <profame>. 7.8 PPH :MOX-NO> EXHAUST CHEMISTRY: 99.5672 % CHEMISTRY:</profame>	Experiment No. 14
TUREINE CURTICA TUREINE CUMCUST TUREINE COMCUST TUREINE SEGERALISIONS FOR THE REED U.S. NAVY AIR ENGIS	TIME TO THE TOTAL THE STATE THE STATE TO THE STATE THE STATE THE STATE S	THET HIR FPESSURE, PSIGNELLET HIR FPESSURE, PSIGNET HIRET HIR TEMPERTURE, DEG F 324.96 FILE FLOW RATE, LBS/BLO 324.96 FUEL/AIR RATIO .02166 AIR FLOW LOADING FACTOR .1.6963	FUEL TEMPERATUZE=125.4 DEG F	+ BURNER OUTLET TEMPE	### ### ##############################	CENTER ANJULUS 2 1771 14 322		* EXHAUST CHEMISTRY	CO 635 % CO24.46 % O214.5 HG 56.5 PFH NO2 STIOF E NUNBER: 23.0 CO-BUSTIÔN GFFICIENCY, CALCULATED FROM EXPRIST CAFEL AIR PATIO CALCULATED FROM EXPRANST CAFEBARENST	Figure 21 - Test Report of
U. S. ARWT FUELS & UNE LEGERALI LABORATOR. TURBINE FOR THE SENERALITY THINK THE EMULSIONS FOR THE SENERAL OF ELHPINST PARTICULATES *** U.S. MAYT RIP EMULERING CENTER	DST : 6/25 75 COLE TOPR SYSTEM T-63 POWER POINT 1062 PEST FUEL: JPS-P ENULSIFIED WITH SX HATER	HTML TEST CONDITIONS ***** SEREB LAST SCAN 72.36 .26 76.20 72.50 72.453 1.27 524.60 522.82 2.507 .913 2.660 2.490 3.234 .029 3.210 3.266 .02151 .00025 .02120 .02166 1.0869 .0076 1.0940 1.0767	FUEL PPESSURE* 0 FUEL TEMPERATURE*121.1 DEG F	. DURINER GUILET TEMPERATUZE SURVEY .			11 1639 12 14 1774 18 10 10 6 6 19 12 626 26 13 1798 13	• EXHAUST CHEMISTRY •	CF 333 % CO24.52 % O214.6 % UBH 3.4 PPH (PROPRIE) HD 51.7 PC. HOX 39.5 PPH NO 59.5 PPH NO 10.1 PPH (PROPRIE) HDX 59.5 PPH NO 10.1 PPH (PROPRIE) HDX 59.5 PC. HDX 7.8 PPH (HDX-HD) HDX	Figure 20 - Test Report of Experiment No. 13

1. S. ARIN FUELS & LUBRICARIS PROJECTA "LABORATORY TURBINE COMPUSIONS FRANCE IN THE NEBRUCION OF EACH ENULSIONS FOR THE NEBRUCION OF EACH PARTICULATES U.S. HRVY AIR EAGINEERING SHIER	DATC: 6/27/75 CONFUSTOR SYSTEM: T-63 POWER POINT: 160% (TEX: FUEL: JPS-HA EMU.SIFIED MITH ON MATER	INLET BIR PRESSUPE, PSIR AVERAGE STD.DEV. DESIRED LAST SCAN INLET BIR PRESSUPE, PSIR 71.54 76.24	FUEL PRESSURE # 89.7 DEG F	+ BURNER GUTLET TEMPERATURE SURVEY +	468	Figure 23 - Test Report of Experiment No. 16
1. S. BANY FUELS & LURRILHITS FOR LABORATURY TURBINE COMBUSTOF FACILITY STATES FOR THE REDUCTION OF EXHAUST PARTICULATES FOR THE REDUCTION OF EXHAUST PARTICULATES FOR THE RESULTION OF EXHAUST PARTICULATES	GATE: 6.25 75 CURDISTOR SITER: 1-63 FONER POINT: 106X TEST FUEL: JPS-P EMULSIFIED MITH 5% WATER	INLET AIR PRESSURE, PSIA AVERAGE STD.DEV. DESIRED LAST SCAN AVERAGE STAN AVERAGE S	FUEL PRESSURE 0 FUEL TEMPERATURE-125.0 DEG F	* BURNER OUTLET TEMPERATURE SURVEY *	BOT FAVE-1687.1 BEG F BOT FAVE-1687.1 BEG F BOT FATTERN FACTOR = .1157 BOT FATTERN FACTOR = .1157 CENTER ANNULUS 1 1763 21 10 1565 28 10 1771 21 11 1596 177 2 1771 21 11 1596 177 2 1771 21 11 1596 177 2 1822 11 11 1596 177 2 1822 11 11 1596 177 2 2386 8 1822 11 11 1596 177 2 2386 8 1742 18 11 1896 9 119 9 12 9 12 9 119 9 12 9 12 9 12 9	Stracks: 3000 P.S.J. 4% and. Figure 22 - Test Report of Experiment No. 15

NAEC-9	2–11	4	z			<u>.</u>	
AN FOLST	S. J.C. 183		1.857 2.1.28 21.36 2.497 3.888 3.888 1.898		•	UGH. 3.2 PPH PROPANE PPH (MOK-HO) I CHENISTRY: 99.3741 X ITRY: .020775	
			2000 2000 2000 2000 2000 2000 2000 200	•	•	3.2 04-160 1817 1817 1826	No. 18
PESERCH A PCALITY		11 6 CINT: 180% R	1100.0 11.0 12.0 13.0 13.0 13.0 13.0 13.0 13.0 13.0 13		RE SURVEY	7.5 PPH (MOK-HO) 7.5 PPH (MOK-HO) EXHAUST CHENISTRYS	Experiment
LOSE TENEN PESCHOOM LAGGRAPHON CONDUSTOR FACILITY THE RESIDENCE OF LONDON	AIP ENTINEERING CLAIRS	TIME: 11: 0 POMER POINT: OR MATER	7EST CONDITIONS 70.66 1 72.96 1 2.96 1 2.983		TEMPERATU		of
	MAY: A18	STEER T-63	m d	ر ور نو	OURNER OUTLET TEMPERATURE	OT 6VG=1673.3 DEG F OT HOT SPOT! TCB 8 = 1804 DEG F OU IER ANNULUS 1 1784 21 CENTER HANNULUS 2 1572 29 CENTER HANNULUS 2 1572 21 INC. 036 % CO2.4.36 % O2.14 INC. 52.5 I'PH NOX., 60.8 PPH NO2.14 FUEL-MATTER FATIO.CALCULATED FROM EXHAUST REMARKS:	t Report
SHIPH.	U.S. HAVY	n: 1-63 MBKEHULSTI		PRESSUPE 0 TEMPERATURE 90.4 DEG	-	TOPE TO THE TOPE T	5 - Test
	5	6 27 5 5162 8 57651 708L1 JPS-16	RATE RATE RATE RATI	RESSUPE= ENPERATUR		OT AVG-1673.3 DEG NOT HOT SPOT! TCB OT PROTERN FACTOR TO IS	Figure 25
-		2018 2018 400 400 400 400 400 400 400 400 400 40	INLET AND INCET AND AIR FLOM FUEL FLOM PUEL ALD	FUEL PA		DOUTER TOUTER TO	
MTCRZ			1.36 323.84 323.84 3.267 3.267 1.6742	•		3 PPH (PROPANE) 0) 4: 99.5286 %	. 17
: *BORATCR2		.e	22.26 22.36 22.36 22.36 32.36 33.36 33.36 34.66		•	USH. 5.3 PPR PPR (BOX-NO) 1 CHEMISTRY: 9	ment No.
.		Sint 1887	170. DEV. 1. 23 1. 39 1. 30 1.		RE SURVEY		Experiment
Live Tolled Collection	T ENGINEER TO	TING: 91 POHER POI	TEST CONDITION NERRICE STI 71.35 526.68 2.496 3.299 92212 1.0944		* BURNER OUTLET TEMPERATU	Chemistry No. 14.5	eport of
FUELS & L.	U.S. MAVY AIR	BOTE: 6/12/73 CORP. FOR SYLTEM: T-63 7657 FORE: JPG-MR ENUISIFIED WITH	¥	E G F	ER OUTLET	= 1654 DEG F NYEPAGE S 17341 1734 1739 1753 1753 1753 1753 1754 1754 1756 1757 1757 1757 1757 1757 1757 1757	Test Report
S. HOW. FOELS. V. TURBINE.		N: 1-63 NA ENULSI	**************************************	E=120.7 D	+ 866	MALUS 13 8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	re 24 -
4 9	•	57.27.77 70% SN:TE	4444 EXPERIMEN INLET AIR PRESSURE, PSIA INLET AIR TERPERATURE, DGC F FUEL FLOM RATE, LBS/SEC FUEL FLOM RATE, LBS/NIN FUEL AIR RATIO AIR FLOM LORDING FACTOR	PPESSURE" 0 TEMPERATURE=120.7 DEG		OUT FOR THE NAME OF TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TOO BE FOUND TO	Figure
·. •		Mate: Confe: 7657 Fi	INCET OF FUEL FLEE	FUEL PI		CENTER CENTER COUTTE SHOWER CONTER CONTER CONTER FUELVAN	

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U. S. ARNY FUELS & LUBRICHATY PESERALM LABORATORY TURBINE CONCUSTOR FACILITY	*** FIND OF FUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PARTICALPTES . U.S. MANY AIR ENGINEERING (FITTR)	COMPACTOR SYSTEM 1-63 POWER POINT: 1002 TEST FACE: JPS-P 02 MATER	***** EXPERIMENTAL TEST CONDITIONS ***** ******************************	FUEL PRESSURE 305 FUEL TEMPERATURE 137.5 DEG F * BURNER OUTLET TEMPERATURE SURVEY +	######################################	
Contractions of the contraction	AND CONTRACTOR OF STATEMENT OF THE STATE	DATE: 6 27 75 CARBUSTOR SISTEM T-63 FOMER POINT: 100: 125T SUBL: JPS-HBRETHULSTFIED MITH 10'S MATER		FUEL PRESSURE* 0 FUEL TEMPERATURE=121.1 DEG F * BURNER OUTLGT TEMPERATURE SURVEY *	## ## ## ## ## ## ## ## ## ## ## ## ##	

PAS FUELY & LUBRICATION OF MACHINE LANGUAGES	TURBINE COMPUSTOR FACILITY
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--- STUDY or FUEL EMBLSTONS FOR THE REDUCTION OF EXAMOSE PARTICULATED U.S. MAYY RIR ENGINEEROOL LENTER

TIMET 10: 1 POWER POINT: 1607; S. WATER BATS: 6-17-76 CARBULTOR SISTEM: T-63 TEST FUEL: JPS-P EMULSIFIED NITH

CAST SCAN 67.20 522.38 2.311 2.899 . 62091 26.5.18E. 524.43 62.43 62.946 62.966 62.966 62.966 63.966 63.966 \$10.00 ***
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\$1 ***** EXPERINENTAL INLET AIR PRESSURE, PSIA INLET AIR TENPERATURE, BEG F AIR FLOW RATE, LBS/SEC FUEL FLOW RATE, LBS/HIN FUEL AIR RATIO AIR FLOM LOADING FACTOR

FUEL PRESSURE= 332 FUEL TENPERATURE=135.3 DEG F

* BURNER OUTLET TEMPERATURE SURVEY

1316 1316 NVERAGE ST 1780 1730 807 8/6=1713.9 BEG F 807 HOT SPOT: 1CB 2 = 807 PATTERN FACTOR = 1CB INNER PRINTLUS **DUTER HARRILUS** CENTER

FFM <PPOPRIE> 99.4618 02.14.7 % UBH.. 6.1 | N02.. 9.0 PPH (N0X.NO) C02..4.37 % NOX.. 55.5 PPH

* EXHAUST CHEMISTRY

Figure 28 - Test Report of Experiment No.

U. S. ARMY FUELS & LUBKICATI, ANSERTON LABURATOR.

** THUS OF FUEL EMULSIONS FOR THE PERICETOR OF EXHAUST OF TIL ALCOSE NOVY AIR ENGINEERING CENTER

TIME: 10:32 FOMER POINT: 180% 10% WATER FETE: 6-17-76 CONBUSTOR SYSTEM: T-63 TEST FIRE: JPS-P EMULSIFIED MITH

66.90 525.62 2.336 2.336 3.882 3.882 52195 1.6961 AVERAGE STD. DEV. 66.86 15.05 524.62 1.34 2.322 .813 3.183 .823 .9227 .88623 1.6968 .8659 24.52 24.52 24.52 24.52 25.53 **** EXPERIMENTAL INLEI MIR PPESSURE, PSIMINLEI MIR TENPERATURE, DEG F MIR FLOII RMTE, LBS/SEC FUEL FLOW RATE, LBS/MIN FUEL/MIR RMTIO AIR FLOW LOADING FACTOR

FUEL PRESSURE 347 FUEL TEMPERATURE=135.3 DEG F

* BURNER OUTLET TEMPERATURE SURVEY

607 896=1724.0 DEC F 501 HOT SF01: TC0 2 = 801 PATTERN FACTOR = TC0 PNACUS CUTER ANNULUS CENTER INNER

PEN PROPRINE 02..14.8 % UBH. 8.5 NO2. 8.8 PPM <NUX-NO CHEMISTRY . C02..4.45 % NOX.. 51.5 PPH * EXHAUST 10. . . 045 % NG. . 43.5 PPH SMOKE NUMBER

22 Figure 29 - Test Report of Experiment No.

7

ARMY FUELS & LUBRICANTS RESEARCH LABURATORY THEN THEN THE COMMISTOR FROM 111Y
RESEARCH
LUBRICANTS
FUELS L
RRHY
è.

*** STUB'S OF FUEL ENULSIONS FOR THE REDUCTION OF EXHAUST PARTICIA, ATES 4 U.S. MANY AIR ENCINEERING CENTER

THE: 6/17/76 THES POWER POINT: 100X POWER POINT: 100X PEST FUEL: JPS-P EMULSIFIED MITH 15% LATER

FUEL PRESSURE= 360 FUEL TEMPERATURE=133.6 DEG F . DURNER OUTLET TEMPERATURE SURVEY

CO. . 044 % COZ. 4.55 % OZ. 14.7 % UBH. 5.4 PPH <PROPRIES SHOKE WARREN 17.6 ST. 52.3 PPH NOZ. 6.3 PPH <NOX-NO; SPACKE NAMBER: 17.6

EXMUST CHENISTRY +

CUMBUSTION EFFICIENCY, CALCULATED FROM EXHINST CHEMISTRY: 99.4976 AFRICAL CALCALATED FROM EXHIUST CHEMISTRY: .021639 AFRICALS AFRICALS:

Figure 30 - Test Report of Experiment No. 23

WHEE WENCE

U. S. ARMY FUELS & LUBRICANTS PESCARCH LABORATORY TURBINE COMBUSTOR FALLLITY

*** STUDT OF FUEL ENULSIONS FOR THE REDUCTION OF EXHAUST PARTICLENTES U.S. NAVY ATP. ENGINEERING CENTER

DATE: 6/17/76 T-63 POMER POINT: 100% CUMBUSTOR SYSTEM: T-63 POLITH 20% WATER

FUEL PRESSURE= 370 FUEL TEMPERATURE=135.3 DEG F

* BURNER OUTLET TEMPERATURE SURVEY *

BOT AVG=1712.3 DEG F
EOT HOT SPOT: TC6 2 = 1833 DEG F
BOT PATTERN FACTOR = 1819

OUTER FANNULUS 1 1794 33

CENTER FANNULUS 2 1833 11

CENTER FANNULUS 2 1833 11

CENTER FANNULUS 2 1833 11

CHARER FANNULUS 3 1768 12

IMMER FANNULUS 3 1829 13

EMPRES FANNULUS 3 1829 13

COMBUSTION EFFICIENCY, CALCULATED FROM EXAMUST CHEMISTRY: 90.3629 FUEL, AIR RATIO, CALCULATED FROM EXAMUST CHEMISTRY: .000732 REMMEKS:

Figure 31 - Test Report of Experiment No. 24

FUELS & LUBPICANTS RESERVE LABORATORY TURBINE COMBUSTOR PACILITY
RESERVED SPECIAL SPECIAL SPECIAL STATES
LUBP 1CHMTS COMBUSTOR
FUELS & TURBINE
. PRRTY

EMM.SIONS FOR THE REDUCTION OF EMHNUST PARTICULATED U.S. MANY AIR ENGINEERING TENTER THE STUDY OF FUEL

TIME: 12:10
POWER POINT: 100%
25% HEMMEL.WATET . E. 6-17-1. LEBUSTOR SYSTEM: T-63 TT-1 FUEL: JPS-P EMUSIFIED MITH

CAST SCAN 65.20 524.:1 2.327 3.449 .62478 DESIRED 65.40 524.60 2.340 3.520 1.5246 1.6946 **** EXPERIMENTAL INLET FIR PRESSURE, PSIA HALET AIR TEMPERATURE, BEG F AIR FLON RATE, LBS/SEC FUEL FLON RATE, LBS/MIN FUEL/AIR RATIO AIR FLON LOADING FACTOR

. BURNER DUTLET TEMPERATURE SURVEY FUEL PRESSURE= 390 FUEL TEMPERATURE=134.5 DEG F

BCT AVG=1702.3 BEG F = 1831 BEG F 801 AOT \$7001 TC0 3 = 1805 BOT PATTERN FACTOR = .1805 AVERAGE \$^{**} \text{control of the pasteurs} 1 1706 AVERAGE \$^{**} \text{control of the pasteurs} 1 1706 CENTER MINICUS THE MARKING

CO2..4.35 x 02x.14.7 x U8H.,14.7 PPN <PROPRNE> NOX.. 38.3 PPH NO2.. 8.8 PPH <NOX-NO> 12.2 ■ EXHAUST CHEMISTRY >

COMBUSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 90.2645 FUEL/AIR RATIO, CALCULATED FROM EXHAUST CHEMISTRY: .021063 REMARKS:

- Test Report of Experiment No. 25 Figure 32

MARINE WARRY

TIME: 9:19 POWER POINT: 1882

0015: 6/18/76 CONSUSINE SYSTEM: T-63 TEST 1 UEL: JPS-P

LAST SCAW 57.66 526.92 2.335 2.748 2.748 1.8856 1.8 52.46 22.346 3.666 .82576 esired TEST CONDITIONS *
RVERACE STD. DEV
567.94 1.0
2.336 .01
2.336 .01
2.336 .00
1.0948 .000 **** EXPERIMENTAL INLET GIR PRESSURE, PSIA INLET BIR TEIPERATURE, DEG F AIR FLOW RHIE, LBS/SEC FUEL FLOW RHIE, LBS/MIN FUEL AIR RHIIO AIR FLOW LUADING FACTOR

* BURNER OUTLET TEMPERATURE SURVEY * FUEL PRESSURE" 320 FUEL TEMPEPATURE"133.6 DEG F

1839 PEG F 1139 PEG F 1739 PEG F 1536 PEG F 1536 PEG F 1532 PEG F 1532 PEG F 1532 PEG F 1753 PEG BOT AVC=1714.6 DEG F BOT HOT SPOT: TC# 2 BOT PATTERN FACTOR = CELITER ANNULUS INNER ANNULUS OUTER ANNULUS

PPE (PROPERE) 02.15,2 % UGH.3.6 \ N02.16.2 PPH < N0X-N0 C02..4.32 % NOX.. 60.5 PPH CO. . . . O39 X NO. SC. 3 PPM SHOILE HIPHBER:

* EXHAUST CHEMISTRY *

8.5348 CUBBUSTION EFFICIENCY. CALCULATED FROM EXHAUST CMEMISTRY: .608882 REMANS RATIO.CALCULATED FROM EXHAUST CHEMISTRY: .608882

Figure 33 - Test Report of Experiment No. 26

LABOF	
ALTERECH	
FUELS & LUCKICANTS RESPONDED THE SECURITY	
FUELS &	
S. ARHY	
3	

A STAUN OF BUEL ERMISIONS FOR THE REBUSTION OF EXNAUST THE LAGGER TO BUST BY AND ENGINEERING CENTER

DHIST 6/8/75
COMBUSTOR SYSTEMS T-63
FONEN POINT 100X
TEST PIELS JPS-P EMULSIFIED HITH 5% HATER

FUEL PRESSURE= 340 FUEL TEMPERATIOE=134.9 DEG F . BURNER OUTLET TEMPERATURE SURVEY

601 6VC=1724.8 BEC F 801 BEC F 801 HUT SP01: TC0 2 = 1841 BEC F 801 HUT SP01: TC0 2 = 1841 BEC F 801 HUT SP01: TC0 NVERNCE STD BEV 91756 GO F 1536 F

• EXMANDSTRY •
(C). . .046 % CO2..4.40 % O2..14.8 % UGH.. 6.3 PP:1 . - P:0FFWE PHD.. 46.3 PPH NOX.. 55.0 PPH NO2.. 8.7 PFM (NOX-NU SHO)E HUMBER: 22.3

UCHBUSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 99.4528 FUEL/AIR MATIO-CALCULATED FROM EXHAUST CHEMISTRY: .eziede PENHENCS: Figure 34 - Test Report of Experiment No. 27

U. S. MRHY FUELS & LUBRICANTS REJERRON LABORATORY TURBINE COMBUSTOR FACILITY

TODY OF FUEL ENALSIONS FOR THE REDUCTION OF EXAMOST PARTICULATES ** U.S. MAYY AIR ENGINEERING CENTER

UNTE: 644676 COMBUSTOP SYSTEM: T-63 POWER POINT: 100X 1ES: FUEL: JPS-P EMULSIFIED WITH 30% WATER

FUEL PRESSURE 415 FUEL TEMPERATURE=125.0 DEG F * BURNER OUTLET TEMPERATURE SURVEY

* EXMANST CNEMISTRY * CO2..4.30 % USM.. 30.8 PPH <PROFINES 1-0.. 26.3 PPH NOX.. 34.4 PPK NO2.. 8.1 PPH <NGK-ND SHOKE NUMBER! 13.9

CONBUSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 90.1670 FUEL AIR RATIO-CALCULATED FROM EXHAUST CHEMISTRY: .03645

Figure 35 - Test Report of Experiment No. 28

P++ STUDY OF FUEL ENULSIONS FOR THE REDUCTION OF EXHAUST PAPTICL, FTE) +- U.S. HAVY FIR ENGINEERING CENTER

EALE: 6/12/78
CGGGSTOR SYSTEM T-63
POWER POINT: 100%
TEST FUEL: JPS-P EMULSIFIED WITH 40% WATER

| INLET HIR PRESSURE, PSIA | AVERAGE STB.DEV, DESIRED LAST SCOME INLET HIR PRESSURE, PSIA | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65.46 | 65

FUEL PRESSURE= 445 FUEL TEMPERATURE=122.8 DEG F * BURNER OUTLET TEMPERATURE SURVEY *

CG...673 % CG2.4.33 % G2.15.6 % UBH.. 30.7 PP': CPRCPANE> NO... 20.3 PPH NOX... 29.5 PPH NOZ... 9.0 PPH CNOX-NO> NO... 20.3 PPH NOX... 29.5 PPH NOZ... 9.0 PPH CNOX-NO> NO CONCPATION EXPANSIT CHEMISTRY: 90.0028 FUCL. ATR KATIO.CALCULATED FROM EXHAUST CHEMISTRY: 90.0028 PERMPKS:

* EXHAUST CHEMISTRY *

Figure 36 - Test Report of Experiment No. 29

WATER WATER LOOP

LABORATORY
FACILITY
LUBRICANTS COMBUSTOR
FUELS & TURBINE
ARH:
ë ë

*** STUBY OF FUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATES

JATE: 6.18 %
COMBUSTOR S.CSTEM: 7-63
FOWER POINT: 25X
TEST FULL: JPS-P
***** EXPERIMENTAL TEST CONDITIONS ####

FUEL TEMPERATURE=157.4 DEG F FUEL TEMPERATURE=157.4 DEG F • BURNER OUTLET TEMPERATURE SURVEY •

EXMANST CHEMISTRY #

CO. . 856 % CO2..2.54 % O2..17.5 % UBH. 140.0 PPN <PROPANE>
NO. 11.7 PPN

MOX. 15.7 PPN

MOX. 19.3 PPN

MOX. 15.7 PPN

MOX. 19.3 PPN

MOX. 17.5 % UBH.

MOX. 10.0 PPN <PPN

FOR CHOMETING

FOR CHARTED FROM EXMANST CHEMISTRY: 96.7223

REHIRKS:

•

Figure 37 - Test Report of Experiment No. 30

11, S. MRMY FUELS & LUBRICANTS PESENRCH LMBORATOPY TURBINE COMBUSTOP FACTLITY +** STUDE OF FUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATES - + U.S. MANY AIR ENGINEERING CENTER	TOPIE: 6.19.76 CONDUSTUR S STEM: T-63 POWER POINT: 252 TEST FUEL: JPS-P EMULSIFIED WITH 203 WATER	++++ EXPERIMENTAL TEST CONDITIONS +++++ HVERAGE STD.DEV. DESIRED LAST SCAN HILLT HIP PFESSURE, PSIA 44.57 .18 45.18 44.68 HILLT RIP PFESSURE, DEG 7 351.81 .49 353.80 352.87 HIR FLOW RATE, LBS/SEC 1.805 .1816 1.805 FUEL FLOW PATE, LBS/MIN .0144 .00014 .01450 .01447 AIR FLOW LOADING FACTOR 1.536 .0681 1.1458 I.1532	FUEL PRESSURE 200 FUEL TEMPERATURE 148.3 DEG F	* BURNER OUTLET TEMPERATURE SURVEY *	807 AVG=1113.9 DEG F 807 HOT SPOT: TCB 3 = 1239 DEG F 807 HOT SPOT: TCB 3 = 1239 DEG F 807 HOT SPOT: TCB 3 = 1239 DEG F 807 HOT STOR = 1645 1 1136 11 1 1016 9 1 1007 7 1 1016 9 1 1005 9 1 1006 9 1 1006 9 1 1006 9 1 1006 9 1 1006 9 1 1006 9 1 1006 9 1 1005 9 1 1006	◆ EXHAUST CHENISTRY ◆	3 x UBH, 327.7 PPN <propane.< th=""><th>CONSUSTION EFFICIENCY, CALCULATED FROM EXMANST CHEMISTRY: 94,9669 PT FIG. GALCULATED FROM EXMANST CHEMISTRY:</th><th>Figure 39 - Test Report of Experiment No. 32</th></propane.<>	CONSUSTION EFFICIENCY, CALCULATED FROM EXMANST CHEMISTRY: 94,9669 PT FIG. GALCULATED FROM EXMANST CHEMISTRY:	Figure 39 - Test Report of Experiment No. 32
U. S. PRNY FUELS & LUSKICANTS RESERRCH LMBORATORY TURBINE COMBUSTOR FACILITY **: TUBY OF FUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATES ** U.S. MAYY AIR ENGINEERING CENTER	DAIE: 5/18/76 COMAUSTOR STSTEN: T-63 POWER POINT: 25x TES! FUEL: UPS-P EMUKSIFIED WITH 18% WATER	INLET AIR PRESSURE, PSIA 44.84 .18 45.10 44.70 18.ET AIR TEHPERATURE, DEG 70.05 .18 45.10 44.70 18.ET AIR TEHPERATURE, DEG 75.10 .61 352.69 352.59 18.ET AIR TEHPERATURE, DEG 751.75 .61 353.60 352.59 18.ET FLOW RATE, LBS/RIN 1.418 .019 1.449 1.413 FUEL/RIR RATIO 1.40 1.413 .01305 RIR FLOW LOADING FACTOR 1.149 1.4130 1.1505	FUEL PRESSURE= 192 FUEL TENPERATURE=152.6 DEG F	+ BURNER OUTLET TEMPERATURE SURVEY +	DOT AVG-1127.5 DEC F DOT HOT SPOT: TCD 3 = 1254 DEC F DOT HOT SPOT: TCD 3 = 1254 DEC F OUTER MANULUS 1 1162 10 T 1025 6 T 1025 10 T 1026 11 TOWER MANULUS 2 1246 11 TOWER MANULUS 3 1254 12	* EXMINST CHEMISTRY *	CO 122 % CO22.50 % 0217.3 % USH, 251.5 PPH (PRUPAME) MI. 12.3 PPH NOX 17.0 PPH M02 4.7 PPH (MOX-NO) STOYE MUNERS: 8.6	COMPISTION EFFICIENCY, CALCALATED FROM EXPANST CHEMISTRY: 96.0419 FALL-WIR MATIO-CALCALATED FROM EXPANST CHEMISTRY: -012923 RESPONS:	Figure 38 - Test Report of Experiment No. 31

i	JRHY	MRHY FUELS &	& LUERICANTS RESEARCH	PESCARCH	LAGORATIES
		TURBINE	COMBUSTOR P	ICICITY	

*** STUD. OF THE EMULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATES ... U.S. MAYY AIR ENGINEIRING CENTER

DIVE: 6/18/76 CONDISTOR SYSIEM: 1-63 POWER POINT: 25% TEST FUEL: JPS-P ENCLSIFIED NITH 30% WATER ### EXPERIMENTML TEST CONDITIONS +****

INEET RIF PRESSURE, PSIA

A.90 .17 44.80

11.4.ET RIP TEMPERATURE, DEG F 352.00

ARE FLOM RATE, L85.5EC 1.801 1.810 1.789

FUEL FLOW RATE, L85.7HH 1.666 .015 1.709

FUEL ARR RATIJ

A.91.432 .00018 .01549 1.1383

FUEL PRESSURE= 205 FUEL TEMPERATURE=141.0 DEG F . BURNER OUTLET TEMPERATURE SURVEY

* EXHAUST CHEMISTRY *
CO. .142 4 CO2..2.40 x O2..17.4 x UBH. 393.7 PPH < PRUPANE. HO. 8.6 PPH NOX. 11.3 PPH MO2. 2.5 PPH < NOX-NO.

COMBUSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 93.0364 FULL-AIR RATIO, CALCULATED FROM EXHAUST CHEMISTRY: .012759 RENAMERS:

U. S. HRWY FUELS & LUBRICANTS ALSEANCH LHBORATSRY TURBINE COMBUSTOR FACILITY

*** STOWN OF FUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PROTICT HISS U.S. NAVY RIR ENGINEERING CENTER

TIME: 146 CHESTOR STORENT T-63 POMER POINT: 25% TEST FUEL: JPS-P EMULSIFIED MITH 40% WATER

fuel pressure= 213 Fuel temperature=139.2 Deg f * BURNER OUTLET TEMPERATURE SURVEY *

BOT AVC-1896.4 DEG F

EUT HOT SPOT: TC# 3 = 1217 DEG F

BOT PATTERN FACTOR = .1720

OUTER PHILLUS 1 1081

CENTER PHILLUS 2 1173

CENTER PHILLUS 2 1173

CENTER PHILLUS 2 1173

CENTER PHILLUS 2 1188

INWER THRULUS 2 1188

INWER THRULUS 3 1217

COMBUSTION FFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 92.1171
FUEL/AIR HATIO.CALCULATED FROM EXHAUST CHEMISTRY: .012047
REHARKS:

Figure 41 - Test Report of Experiment No. 34

Figure 40 - Test Report of Experiment No. 33

NA	EC-	9	2-	1	1
***	-	•		-	-

U. S. ARMY FUELS & LURRICANTS RESERVEN LABORATURY TURZINE UCHBUSTOR FACTLITY	
FUELS & TUR? INE	
HRMY	
-3	

d. S. ARMY FUELS E LACKICANTS RESERVED LABORATION TURBLIE LACKING FROIL ITS

STUDY OF FUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATE! -
JE EXHINST CENTER
REDUCT (**). ENGINEER II'S.
IS FOR THE MANY SIR (
EMULSION U.S.
OF FUEL
STUBY

*** STUBY OF FUEL EMUSIONS FOR THE REDUCTION OF EXHAUST PARTICULATE! - *** TOTAL FRUELSIONS FOR THE EXHAUST PAPTICULATES +** U.S. MAY) BIR ENGINEERING CHITER	TIME: 10:04 POWER POINT: 100X ITH 40°, WATER	+*** EXPERIMENTAL TEST CONDITIONS +*** RESSURE, PSIA 67.82 2.83 52.40 68.40 68.30 68.80 68.40 68.30 68.80 7.80 7.80 7.80 7.80 7.80 7.80 7.80
ANN "SID THE THE STORESTORES FOR	DAMES 6.21.76. CONFOSTOR S.STEN T-63 TEST FUEL: JPS-P ENULSIFIED HITH	INLET AIR PRESSURE, PSIA INLE: AIR TEMPERATURE, DEG F FUEL FLOW RNTE, LBS-SEC FUEL-RPON RATE, LBS-MIN
1	•	_
CULATE:		1.851 SCR. 58.58 \$25.88 \$2.388 \$2.831 \$2.831 \$3.831
MUST PARTICULATE:	ä	DESTRED LA 52.4 % 66.4
tion. De exhiust particulate: Erin. Center	16: 3 Point: 180%	05 SIRED 22.40 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
The Reduction of Exhiust particulates Air Engingeriis, Center	TIME: 10: 3 POWER POINT: 100%	DESTREE 65.40 324.96 2.346 2.346 2.926 3.936

FUEL PRESSURE= 318
FUEL TEMPERATURE=126.7 DEG F

* BURNER OUTLET TEMPERATURE SURVEY *	14.	1	STB DEV	200	ន	•	12	13	17	**	<u> </u>		~	34	13	25	2
. BURNER OUTLE	r 6	\$601	A RVERMGE	1219	7 1680	1559	3 1592	2 1859	5 1773	8 1792	11 1679	1730	3 1846	9291 9	9 1785	2 1652	1796
	BOT 845=1719.1 DEG BOT HOT SPOT: TC#	PATTERN FAC	OUTS AMMERING	•		7	•	CENTER ANNULUS					INHER PARTILUS				
			7		••	٠				, .	-	•	•		-	•-•	• .

CO2..4.35 % 02.,15.1 % UBH..5.5 PPH < PROPRIE: MOX.. 61.7 PPH MO2.. 5.7 PPH < MOX-NO.> 28.5 * EXMANST CHEMISTRY *

JUSTON EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 90.5465 FUEL-AIR MATIO, CALCULATED FROM EXHAUST CHEMISTRY: .020564 MFMORKS:

Figure 42 - Test Report of Experiment No. 35

510.NS ***** 510.DEV. DESIRET LAST SCAN 2.83 524.00 531.45 .018 2.340 5.346 .023 3.978 4.818 .00027 .02770 .02854 .00771 1.0940 1.0017	S SURVEY *	
INLET AIR PRESSURE, PSIA 67.82 INLET AIR PRESSURE, PSIA 67.82 INLET AIR TEMPERATURE, DEG F 526.71 FUEL FLOW RATE, LBS./REC 2.367 FUEL / AIR PRATE, LBS./RIN .02796 AIR FLOM LOADING FACTOR 1.09962 .00	FUEL TEMPERATURE=118.1 DEG F FUEL TEMPERATURE=118.0 DEG F * BURNER OUTLET TEMPERATURE SURVEY	BUT 17'G=1673,4 DEC F BOT HOT SPOT: TC# 2 = 1796 DEC F BOT FRITERN FACTOR = .1023 OUTER AWMULUS 1649 10 13 1582 10 14 1753 12 15 1549 10 15 1582 10 16 1593 12 16 1593 13 16 1593 15 16 1593 15 17 17 18 18 17 18 18 17 18 19 17 18 19 17 18 19 17 18 10 18 18 18 10 18 18 18

CO. . 070 % CO2..4.40 % O2..14.7 % UBH. 25.5 PPH (PROPRIE) NO. . 25.8 PPH (NOX-NO) SHEET NOS. . 6.8 PPH (NOX-NO) SHEET NUMBER: 9.5 LOMENSTION EFFICIENCY, CALCULATED FROM EXHMUST CHEMISTRY: 99.0024 FUEL-RIR RATIO.CALCULATED FROM EXHAUST CHEMISTRY: .031270 REHARKS:

■ EXHAUST CHEMISTRY ◆

Figure 43 - Test Report of Experiment No. 36

A LUDY OF THEL EMULSIONS FOR THE REDUCTION OF EXHAUST PARTM ULLIES NAVY AIR ENGINEERING CENTER

time: 2:25 Fower Puint: 100%

U. S. ARNY FUELS & LUBRICHHIS RESERVER (MBGRATORY	CABINE CONDICTION
PERIOR STREET,	

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• *** TUBY OF FUEL EMULSIONS FOR THE REDUCTION OF ELRICAT PRATICALINES OF U.S. MAYY AIR ENGINEERING JENTER

DATE: 6.21.75 CORRUSTOR SYSTEM: T-63 TEST FUEL: JP5-P EMULSIFIED WITH 50% WATER

LMST SCAN 68.68 327.33 2.373 2.373 4.246 4.246 1.0871 DESIRED 55.40 524.08 2.340 4.260 .02970 1.0940 FST CONDITIONS ***

FVERAGE STD. DEV.

68.39 .17

526.14 2.74

4.239 .021

4.239 .021

1.6826 .06093 ***** ENPERIMENTAL IMET AIR PRESSURE, PSIA IMET AIR TEMPERATURE, DEG F RIR FLOW RATE, LBS/SEC FULL FLOW RATE, LBS/MIN FUEL/AIR RATIO AIP FLOW LOADING FACTOR

FUEL PRESSURE= 470 FUEL TEMPERATURE=114.6 DEG F

* BURNER OUTLET TEMPERATURE SURVEY

- 1763 DEG F - 0829 - 0829 - 1689 - 1788 - 1582 - 1582 - 1583 - 1763 - 1763 - 1763 682 1747 1747 1631 1611 1611 1724 BOT AVC=1668.4 DEG F BOT NOT SPOT: TC# 2 BOT PATTERN FRCTOR = CENTER PHANEUS OUTER HINNELUS STATES PROBLUS

02..14.7 % UBH. 34.9 PPH <PROPRIES H02.. 5.9 PPH <H5K-NO> EXHAUST CHEMISTRY . C02..4.48 % NOX.. 26.4 PPH 8.2

\$4.8921 COREUSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: .021335... FUCL. AIR RATIO, CALCULATED FROM EXHAUST CHEMISTRY: .021335... REMARKS:

37 Test Report of Experiment No. Figure 44

20.00 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	7. 3. 3. 3. 3. 3. 3. 3. 3. 3. 3. 3. 3. 3.	AVERAGE STD. DEV. D 66.16 STD. DEV. D 56.16 3.43 2.310 017 2.716 00023 60022	DESTRED 52.40 52.40 2.280 2.780 2.786	LAST SCAN 66.18 523.68 2.382 2.382 2.382 1.893
---	---	---	--	--

**** EXPERIMENTAL TEST CONDITIONS **** .0078 . 01960 1. 0970 INLET AIR PRESSURE, PSIA INLET AIR TEMPERATURE, DE MAK FLOI KANE, LBS-SEC FUEL FLOW RATE, LBS-MIN FUEL AIR RATIO AIR FLOW LGADING FACTOR

* BURNER OUTLET TEMPERATURE SURVEY FUEL PRESSURE 300 FUEL TEMPERATURE=137.1 DEG F

8 2932 8932 8932 1678 1678 1678 1786 1786 1793 1794 BOT AVG=1682.8 DEG F BOT AOT SPOT: TC# 9 BOT PATTERN FACTOR = CENTER ANNILUS OUTER ANNULUS INNER RNNULUS EXHAUST CHEMISTRY +

PPH < PROPRINE > 02..15.2 % UBH..4.3 | NO2.. 5.7 PPH <NOX-NO> CO2..4.15 % G NOX.. 55.5 PPM 26.6 CC....040 X NG.. 49.8 FFM SMOKE NUMBER:

COLLISTION REFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 90.5010 FULL AIR RHIIO, CALCULATED FROM EXHRUST CHEMISTRY: .01900 RELIAPKS:

38 Figure 45 - Test Report of Experiment No.

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S. ARMY FUELS & LUSKICANTO FLUCKPON LABORATURY TURBINE CUNBMSTOP FATILITY
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*** STUD. OF FUEL EMPLSIONS FOR THE REDUCTION OF EXHAUST PRATICULATES **	U.S. MAY: AIP ENGINEERING CENTER
UF FUEL	
S196.	

186%	
2:51 OIMT:	ند
TIME: 2:51 POWER POINT:	23.
	=
	15 15 1
-	2
ZaTz: e/21-76 Contysiük System T-63	DAY-IND
8.78 30.68	
Sele:	1

_	Z	2				 %	
	LAST S	65.0	526.	2.3	3.1	. 022	1.1084
•	DESIRED	65.40	524.00	2.280	3.150	. 82288	1.0940
ITIONS ****	STD. DEV.	. 24.	2.89	. 619	.826	.08826	1.1947 .8889
TEST COND	AVERAGE	65.56	527.36	2,365	3.165	. 02289	1.1947
**** EXPERINENTAL							AIR FLUH LOGDING FACTOR

FUEL PRESSURE" 343 FUEL TEMPERATURE=130.6 DEG F

* BURNER OUTLET TEMPERATURE SURVEY *

T 22	82221	122828	122
= 1777 DEG .0794 RVERAGE 1678	151 151 157 150 150 150 150 150 150 150 150 150 150	1777 1691 1691 1718	1663 1639 1727
BOT A'G=1685.5 BEG F BOT HOT SPOT: TC# 8 BOT PATTERN FACTOR * TC# OUTER FANALUS 1	6 .	11 11 14 IANGER RINNALUS 3	. นีซี.

. EXMAUST CHEMISTRY .

CO... 056 % CO2..4.20 % O2..14.9 % UBN..14.3 PPH (PROPRNE) SMO... 35.8 PPH NOX.. 41.5 PPH NO2.. 5.7 PPH (NOX-NO) SMOKE NUMBER: 13.2

COMBUTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: DELABOR FUEL AIR RATIO, CALCULATED FROM EXHAUST CHEMISTRY: DEBISE REMARYS!

U. S. ARMY FUELS & LUBPICHMIL NEWERNCH LABORATURY TURBINE COMBUSTON FACILITY

UNC UN TUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PRETICULATES ** U.S. NAVY AIR ENGINEEPING (BITER

DATE: (21/76) FOREN 1-63 POWEP POINT: 180% TEST FUEL: JPS-HBR EMULSIFIED WITH 30% WATER

	LAST SCAN 66.10 66.10 223.4 2.53.5 3.523 .02536 1.8992	
1	DESTRED 65.40 524.90 2.280 3.560 62570 1.0940	
TEST CONDITIONS ++**	310. 054. 3.18 3.18 .00.030 .60.031	
TEST COND	7VERBER 655.91 22.304 3.522 0.522 1.02549 1.0984	
***** ENPEPINENTAL	ILLET AIR PRESSURE, PSIA INLET AIR TEMPERATURE, DEG F AIR FLOW PATE, LBS/SEC FUEL FLOW PATE, LBS/NIN FUEL AIR RATIO AIR FLOW ! OADING FACTOR	

fuel Pressure= 378 Fuel Temperature=123.3 deg f * BURNER DUTLET TEMPERATURE SURVEY *

601 F-G-1650.9 DEG F BAT HOT SPOT: 7C# 8 = 1726 DEG F BOT FATTERN FACTOR = .0672 OUTER RNHULUS I 1639 LEHTER ANNULUS Z 1724 IZ CENTER RNHULUS Z 1724 IZ 1 1664 I3 INHER RNHULUS S 1636 I0 INHER RNHULUS S 1636 I0 INHER RNHULUS S 1636 I0 INHER RNHULUS S 1630 I0

. EXHAUST CHEMISTRY .

CO. . 069 2 CO2.14.15 % 02.14.9 % UEM. 28.6 PPH :PRUPRINE NO., 26.8 PPH NOX., 33.0 PPH NO2., 7.0 PPH (NOX-NO) THATE NUMBER: 9.3

LCGENSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 99.0190 FUEL'AIN RATIO.CALCULATED FROM EXHAUST CHEMISTRY: .020219 Reharks:

Figure 47 - Test Report of Experiment No. 40

Figure 46 - Test Report of Experiment No. 39

. S. APP	FAR' COLATE: TING G FUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATES U.S. MANY AIR ENGINEERING CENTER	TIME: 413 CONSUCTOR SYSTEM: T-63 POMER POINT: 75% TEST FUEL: JPS-HBR EMULSIFIED MITH 15% WATER	LRST SCAN 477.27 477.27 27.27 27.29 618.7 1.0994 FUEL F 1.0994 ARR FL	ic	* BURNER OUTLET TEMPERATURE SURVEY *	DEG F		11 1435 17 14 1485 17 14 1485 17 13 1487 17 9 1374 19 12 1393 19 15 1516 14	* EXHAUST CHEMISTRY *	H. FROPPHE CU 099 X CD23.48 X D216.8 X UBH. 89.0 PPH (PROPANE) HO 23.0 PPH NOX 29.3 PPH HOZ. 6.3 PPH (HOX-NO) CHOKE HIMBER: 12.3 CHOUNTED FROM EXHAUST CHEMISTRY: 97.9108 FULL FIRE RATIO.CALCULATED FROM EXHAUST CHEMISTRY: 97.9108 REMARKS:	
ý á	*** STUD) OF FUEL ENULSIONS FOR THE REDUCTION OF EXHAUST FAR' U.S. MAYY AIR ENGINEERING CENTER	DATE: 6. 21/76 CANGUSTOR SISTEM: T-63 POHER POINT: 75% TENT FUEL: JPS-HBR	***** EXPERIMENTAL TEST COUDITIONS ****** INLET AIF PRESSURE, PSIA SA24 .65 57.40 INLET AIR TEINFERTURE, BEG F 3.65 472.80 RIR FLOW RAIE, LBS/SEC 2.098 .018 2.110 FUEL FICH MATE, LBS/HIN 2.183 .031 2.160 FUEL AIR RATIO .031 2.160 AIR FLOW LOWING FACTOR 1.1813 .0159 1.1238	FUEL TEMPERATURE=154.8 DEG F * BURNER OUTLET TEMPERATURE SURVEY *	AVG=1483.2 DEG F	# 1602 DEG F . 1174 RVERAGE STB 1510 1474	11111111111111111111111111111111111111	= 1 %193	* EXPROS CHEMISTRY *	TOTAL - NO. 29.4 PPH NO. 39.8 PPH NO. 29.5 PPH CANN-NO. 39.8 PPH NO. 29.4 PPH CANN-NO. 39.8 PPH NO. 29.4 PPH CANN-NO. 39.8 PPH NO. 29.6 PPH CANN-NO. 59.8 PPH NO. 29.6 PPH CANN-NO. 50.5 PPH NO. 29.6 PPH CANN-NO. 50.5 PPH CANN-NO. 50.5 PPH NO. 50.5 PPH N	

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ATC.
U. S. APIN FUELS & LUBPICAINS RESENACH LABORATOR TURBINE CONFUSICE FACILITY
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S. ARMY FUELS & LUSTICATION RESERVEN LABORATORY TURBINE CONSUSTUR FACILITY

TIME: 3148 POWER POINT: 55%

UATE: 6/22/76 COMBUSTOR SYSTEM: 1-63 TEST FUEL: JPS-HBR

R THE PEDUCTION OF EXHAUST PARTICULATES ** STUDY OF FUEL ENULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATES ** D.S. HAVY BIR ENGINEERING CENTER
U.S. NAVY AIR ENGINEERING CENTER U.S. NAVY AIR ENGINEERING CENTER

4	ž
	TIME: 9:16 Poher Point: 30: Mater
	UNIE: 6.22.78 CUMBUSCUS SYSTEM: T-63 TEST FUEL: JPS-MBR EMULSIFIED MITH

Supply and a supply as	***************************************				
THENEWS EXTENSIONED	TITAL CONTIN				
	AVERAGE ST	D. DEV.	DESIRED	LAST SCAN	THE BIK PRESSIRE, PSIA
	57.33	.16	57.48	57.63	THLET HIP TEMPERATURE, DEC F
.	471.98	1.19	472.88	473.69	AIR FLOW RATE. 185/SEC
	2.839	915	2.118	2.16.2	FLIEL FLOW BATE, LECAMIN
FUEL FLOW RATE, LOS-MIN	2.729 .020	020	2,736	2.730	FUEL AIR RATIO
	. 02177	. 98619	82168	. 62182	ATR FLOW LOSDING FACTOR
	1.1127	.0083	1.1230	1170	

PRESSURE, PSIA 50.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 . 36. 37.92 .

FUEL PRESSURE= 210 FUEL TEMPERATURE=137.1 DEG F

FUEL PRESSURE" 285 FUEL TEMPERATURE=124.1 DEG F

URVEY *

3									
* BURNER OUTLET TEMPERATURE SI		STD DEV	3 E		22	9:1	e 51 &	33	
	•	S							
3. OFT	22 DEG	AVERAGE	208 0	333 355	618 472	419	1495 1622 1388	364	Ī
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	DEC 100	<u> </u>		~~		~ ~ ·	.	-	-
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	AVG=1465.	PRINTE			CENTER ANNULUS		INER ANNULUS		
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C02.3.60 % 02.16.1 % UBH. 101.5. PP1: (PROPNIE) NOX. 25.0 PPH N02. 4.5 PPH (H0X-MO) 7.6 * EXMINST CHEMISTRY . CO. . 104 ? HG. . 20.5 FPH Shiff NUMBERS

CONSUSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 97.0007 FUEL. JIR RATIO-CALCULATED FROM EXHAUST CHEMISTRY: .01720 PENANKS:

Figure 50 - Test Report of Experiment No. 43

						NA
	·				PPH (PRUPANE)	94.0293
					.8 PPH < HOX-140	EXMAUST CHEMISTRY: CHEMISTRY: .015122
STD DEV	 	.252		CHEMISTRY .		EXHIUST CHE
	1316	1208 1208 1206 1216	1165	. EXHAUST	02 3.6 7 % 0% 3 6 .3 PP	NCY, CALCULA CULATED FROM
BOT HVG=1275,8 DEG BOT HOT SPOT: TCB BOT PATTERN FACTOR TCB	OUTER HANDLUS 1	CENTER HINULUS 2	INNER ANNULUS 3	•	CO 684 2 C NO 22.5 PPH W SMOKE NUMBER: 19.0	COMBUSTION EFFICIENCY. CALCULATED FROM FUEL AND FUEL AND RAFIO. CALCULATED FROM EXHAUST REMARKS:
	DEG F TC0 2 = 1482 DEG F CTOR = .2433 TC0 RVERAGE STD	HVC=1275, 8 DEG F HVT SPOT: TC0 2 = 1432 DEG F HVT SPOT: TC0 2433 TC0 RVERNGE STD TER ANNULUS 1 1419 1 1316 1 20 1120 13 1230	HVG=1275.8 DEG F HOT SPOT: TCB 2 = 1452 DEG F PATTERN FACTOR = .2433 TCB RVERGE STD TCB RVERGE STD 7 8 1316 13 1238 TER HINNULUS 2 1288 6 1296 6 1296 7 13 1238 TER HINNULUS 2 1288 6 1296 7 14 1316 7 14 1316 7 14 1316 7 14 1316 7 14 1317	HVG=1275.8 DEG F HVOT SPOT: TCB 2 = 1452 DEG F PATTERN FACTOR = .2433 TCB RVERAGE STD 1419 1 1419 1 1419 1 1419 1 1230 1 1230 1 1482 1 1230 1	HVC=1275.8 DEG F HVOT SPOT: TCB 2 = 1452 DEG F PATTERN FACTOR = .2433 TCB RVERGE STD 1419 13 1230 128 1482 1582 1682 1682 1181 1683 1683 1683 1683 1683 1683 1683	FOGE 1275.8 DEG F HOT SPOT: TC0 2 FPATTERN FACTOR = 10 13 TER HINNULUS 2 13 TER HINNULUS 2 13 TER HINNULUS 3 11 11 11 12 122:5 PPH NOX

Figure 51 - Test Report of Experiment No. 44

ES & LUCKITANTS EES ROINE COMBUSTOR FRUIT S FOR THE REDUCTION TIME: 10:4 POWER POINT ENTAL TEST CONDITION F 429-57 1.344 1.1344 F 1.344	# BURNER OUTLET TEMPERATURE SURVEY # BOT AVG=1248.0 DEG F BOT HOT SPOT: 764 2 = 2455 BOT PATTERN FACTOR = 2455 OUTER ANNULUS 1 1399 12 CELITER ANNULUS 2 1449 12 CELITER ANNULUS 2 1449 12 S 1879 77 S 1879 9 IMMER ANNULUS 3 1397 15 IMMER ANNULUS 3 1397 15	EC121 % CO23.05 % O216.7 % UBH. 194.3 PPH FEOPPHE. NO 15.3 PFH FOR FEOPPHE. NO 15.3 PFH FOR FOR FOR FOR FURE. 5.9 FOR SHOWE STORE FOR EXHRUST CHEMISTRY: 94.4240 FUE: BIR RATIO.CALCULATED FROM EXHRUST CHEMISTRY: .01224 REMHINKS: Figure 53 - Test Report of Experiment No. 46
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7.257.4 7.257.4 1.397.6 1.231.4 1.231.4 1.189 1.189 1.189 1.189 1.189 1.189 1.189 1.189 1.189

CENTER PRODUCUS

INTER PRINCIUS

02.16.5 % UBH.156.0 PFH <FPOPANE>
NO2. 5.3 PPH <NOX-NO>

CD. . . 109 % CO2..3.00 % NO. . 18.5 PPH NOX. . 23.8 PPH SLOKE HUMBER: 9.9

* EXHAUST CHENISTRY *

CURBUSTION EFFICIENCY, CALCULATED, FROM EXMAUST CHEHISTRY: 96.8656 FLELYNIR PATIO, CALCULATED FROM EXMAUST CHEMISTRY: .015154 PERSORYS:

* BURNER OUTLET TEMPERATURE SURVEY *

FUEL PRESSURE: 222 FUEL TEMPERATURE:134.9 DEG F

LAST SCIN 50.80 426.32 1.945 1.911 1.911 1.1399

DESIRED 58.70 439.00 1.950 1.940 1.940 1.940

FEST CONDITIONS *****

NUERAGE SID. DE 79. 82. 86 1.17 42. 86 1.17 4. 818 1.93 8.828 8.818

INLET AIR PRESSURE, PSIA INLET AIR TENPERATURE, BEG F AIR FLOM RATE, LBS/SEC FUEL FLOM RATE, LBS/NIN FUEL/AIR PATIO AIR FLOM LOADING FACTOR

STURY OF "UEL EMULSIONS FOR THE REDUCTION OF EXHIBST PARTICULATES ** U.S. NAVY AIR ENGINEERING CENTER

TINE: 10:22 PONER POINT: 55% 15% WATER

DRIES 6/20/26 CORBUSTOR SYSTEMS T-65 TEST FUELS JPS-WER EMULSIFIED WITH

***** EXPERIMENTAL

19. S. ARMY FUELS & LUBRICARITS RESEMECH LABORATORY TURBINE COMBUSTOR FACILITY

Figure 52 - Test Report of Experiment No. 45

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OUTER MANULUS

AVG=1233.3 DEG F HOT SPOT: TC# 2 PRITERN FRCTOR =

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LABORATORY	
RECEARCH	FRCILITY
FUELS & LUBRICHIES RECERREN	COMBUSTOR
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	TURBE
PP.H.	
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* *** STHEY OF FUSE EMULSIONS FOR THE REDUCTION OF ENHAUST PARTICULATES ** U.S. MAYY AIR ENCINEERING LENTER

TIME: 9:38 POWER POINT: 8% WATER DMIE: 6.23.76 CONCINSION SISTEM: T-63 TEST FUEL: JAD-MON ENULSIFIED MITH

LAST SCAM 45.40 400.85 1.891 1.414 .01309 42.10 397.00 1.796 1.400 1.1630 AVERAGE STB.BEV. 45.75 -43 401.16 -023 1.423 -023 1.1661 -0135 . 43 . 023 . 623 . 623 . 623 . 633 . 633 **** EXPERIMENTAL INLET AIR PRESSURE, PSIA INLET AIR TEMPERATURE, DEG F AIR FLOW RATE, LOS-SEC FUE. FLOW RATE, LOS-NIN FUEL-AIR RATIO AIR FLOW LOADING FACTOR

FUEL PRESSURE" 190 FUEL TEMPERATURE" 133.6 DEG F

+ BUINER OUTLET TEMPERATURE SURVEY +

BOT RVG-1241.3 BEG F
BOT HOT SPOT: TGB 2 = 1382 BEG F
BOT PRITERN FACTOR = 1676
50 OUTER ANALUS 1 1285
7 1 100
100 100
1100 1100
1231
1331
1332
1231 INER MULUS

02..17.2 x UBH. 112.3 PPN (PPUPANE) 102.. 5.8 PPN (HOX-NO) EXHIUST CHEMISTRY C02..2.76 x MOX... 21.2 PPH 13.3

3.25 COMBUSTION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: - 413637 REHNRIS:

H. S. ARMY FUELS & LUBKICHNIE PESERRIH LABORATOKY TURBINE COMBUSTOR FACILITY

OF FUEL EMULSIONS FOR THE REDUCTION OF EXHBUST PARTICULATES U.S. MAYY AIR CHIGHEPING CENTER λ.

DATE: 6/28/76 COMBUSTOR SYSTEM: 1-63 POWER POINT: TEST FUEL: JPS-HOR EMULSIFIED MITH 15: MATER

15T SCRII 44.80 397.62 1.758 1.688 1.688 1.1782 AVERAGE STD.BEV.

44.19 .28

44.19 .1.17

1.77 .025

1.595 .601

1.1779 .601 . 025 . 025 . 025 . 025 . 025 . 025 **** EXPERIMENTAL INLET AIR PRESSURE, PSIA INLET AIR TEMPERATURE, DEG F AIR FLOW RATE, LBS/SEC FUEL FLOW RATE, LBS/MIM FUEL AIR RATIO AIR FLOM LOADING FACTOR

* BURNER OUTLET TEMPERATURE SURVEY FUEL PRESSURE 200 FUEL TEMPERATURE 134.9 DEG F

1363 DEG 7 1721 DEG 7 1266 ST 1215 DEG BOT AVG=1221.6 DEG BOT HOT SPOT: TCB BOT PRITERN FACTOR CENTER MINULUS **DUTER RHINULISE** INNER MAKEUS

02.17.8 % UBM. 200.0 PPH <PROPRIES HOZ. 4.8 PPH <MCK-NOS . EXHIBS CHEMISTRY . C02..2.76 x HOX.. 16.8 PPH S.2 CC. . 126 x HO. . 12.6 PPH SRV:: RANGER

48 Figure 55 - Test Report of Experiment No.

Figure 54 - Test Report of Experiment No. 47

Martin water in the

* * STILD OF FUEL EMMISIONS FOR THE REDUCTION OF EXHAUST PARTICULATES U.S. MANY AIR ENGINEERING CENTER

OF FUEL EMULSIONS FOR THE REBUCTION OF EXMANS: PARTICULATES . U.S. MAYY AIR ENGINEERING CENTER

190 C ...

TIME: 10:38 POLIER POINT: 402 30% WATER

LANE: 5.28/76 CONGUSTOR SYSTEM: T-63 TEST FUEL: JPS-MOR EMULSIFIED WITH

U. S. ARHY FUELS & LUGRICANIS KESEMECH LABORATORY TURBINE CHRSUSTOR FACILITY

23% TIME: 11:13 POWER POINT:

WATE: C.26/76 COMBUSTOR SYSTEM: T-63 TEST FUEL: JPS-MRR

***** EXPERIMENTAL INLET' AIR PRESSURE, PSIA INLET AIR TEMPERATURE, DEG F AIR FLON RATE, LBS/SEC FUEL FLOM PATE, LBS/MIN FUEL/AIR RATIO AIR FLOM LOADING FACTOR

LAST SCAN 46.30 399.13 1.788 1.843 1.843 1.1317

2.7600

INLEY AIN PRESSURE, PSIA INLET AIR TEMPERATURE, DEG F AIR FLOM RATE, LBS/SEC FUEL FLOM RATE, LBS/MIN FUEL AIR RATIO AIR FLOM LOADING FACTOR

STD. DEV.

**** EXPERITENTAL

DESIRED 353.00 353.00 1.560 1.135 1. AVERAGE STD. DEV. 39-81 .15 .68 .156 .809 .1147 .912 .6121 .69614 .11462 .8682

35. 20 36. 23 1. 360 1. 146 1. 146 1. 146 1. 146

FUEL PRESSURE= 175 FUEL TEMPERATURE=144.8 DEG F

+ BURNER OUTLET TEMPERATURE SURVEY

BOT AVG=1133.2 DEG F EDT HOT SPOTT TC# 3 = 1264 DEG F BOT PATTERN FACTOR = .1685 TC# AVERAGE STD OUTER ANNULUS 1 1162 7 1110

02.17.4 % USH, 163.7 PPH <PROPAHE . H02.. 4.3 PPH <ROX-NO> * EXMAUST CHEMISTRY * CO2..2.52 % NOX.. 15.3 PPM CO. . . 114 % NO. . 11.0 PPH SHOKE NUMBER:

COMENSTION EFFICIENCY, CALCULATED FROM EXHGUST CHEMISTRY: 96.1639 FUNL.AMERRATIO: CALCULATED FROM EXHAUST CHEMISTRY: .012763 Remarks:

. BURNER DUTLET TEMPERATURE SURVEY +

FUEL PRESSURE= 212 FUEL TEMPERATURE=128.9 DEG F

= 1331 BEG | 1770 PVERRUE | 1241 | 1199 | 1104 BOT H'/C=1190.7 DEG F BOT HOT SPOT: TC# 2 BOT PATTERN FACTOR = **OUTER PHANKLUS**

IMEP MERLUS

* EXMAUST CHEMISTRY +

02.17.8 % UBH. 279.3 PPH (PROFANE) WG2. 2.5 PFH (NOX-NO) CO2..2.70 % NOX.. 12.7 PPH CO. . 136 2. HO. . 10.2 PPH SHORE NUMBER:

COMPUSATION EFFICIENCY, CALCULATED FROM EXHAUST CHEMISTRY: 95.0206 FUGIL. AIR RATIO, CALCULATED FROM EXHAUST CHEMISTRY: .013959 KENNIKS:

Figure 56 - Test Report of Experiment No. 49

On Carried

20

Figure 57 - Test Report of Experiment No.

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LABORRTORY
ARMY FUELS & LUSPICANIS RESERREN LABORAY TURBINE CONSCITOR FACILITY
UELS & LUX URBINE CO
. PRENT F
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*** STUDY OF FUEL EMULSIONS FOR THE MEDUCTIUM OF EDMINST PRETICULATES ** U.S. MAYY ATA CHOTHERING CENTER

11ME: 11:38 COMBLEGOR SISTEM: T-63 TEST FUEL: JPS-NOR ENLISTEED MITH 15% MATER

1.550 1.27 1.560 1.27 1.137 NVEFACE - STD. BEV. 39.89 . 14 39.89 . 18 1.567 . 809 1.292 . 821 1.196 . 8005 1.196 . 8005 **** EXPERIMENTAL INLET NIR PRESSURE, PSIA
INLET AIR TEMPERATURE, MEG F
AIR FLOW RATE, LBS/SEC
FUEL FLOW RATE, LBS/NIN
FUEL/AIR RATIO
AIR FLOW LOADING FACTOR

FUEL PRESSURE" 185 FUEL TEMPERATURE=142.3 DEG F

. BURNER OUTLET TEMPERATURE SURVEY .

BOT ATC=1116.6 DEG F BOT NOT SPOT: TC0 3 BOT PRITERN FACTOR * CENTER PHACLUS OUTER MONULUS INCR PHALUS

62.17.5 % UBM.26.7 ' PPR (PROPANE) HJ2., 3.0 PPR (NOX-NO) . Exhaust Cheristin . CO2..2.36 % HCK.. 12.3 PPR CO. . 135 % NO. . 9.3 PPH SPOKE RUNGER: 3.4

CHEFYSTION EFFICIENCY, CALCALATER F.OM EJAKUST CHEHISTRY: 94.4994 FUEL/ALR ANTIO-CALCALATER FROM EXHAUST CHEHISTRY: .012800 MEMBERS:

Figure 58 - Test Report of Experiment No. 51

U. S. ARHY FUELS & LUGHICHNIS RESERVEN LABORATORY TURBINE COMBUSTOR FACILITY

÷ AND SHIDS OF FUEL EMULSIONS FOR THE PERUCTION OF EXHIUST PARTICULATES U.S. MANY AIR ENGINEERING CENTER

23% DDITE: 6/28/76. CUMBULTOR SISTEM T-63 POWER POINT: TEST FUEL: JPS-HOR EMULSIFIED MITH 30% HATER

25.28 25.78 25.75 1.554 1.454 1.554 **** EXPERIMENTAL INLET AIR PRESSURE, PSIA
INLET AIR TENPERATURE, DEG F
AIR FLOM RATE, LBS/SEC
FUEL FLOM RATE, LBS/MIN
FUEL/AIR RATIO
AIR FLOM LOADING FACTOR

FUEL PRESSURE= 185 FUEL TEMPERATURE=139.7 DEC F

* BURNER OUTLET TEMPERATURE SURVEY *

BOT HYG=1882.2 BEG F BOT HAT \$POT: TC# 3 BOT PATTERN FACTOR = CENTER PHINCLUS **OUTER PARTOLUS** INNER PROPULUS * EXHAUST CHENISTRY *

02..17.5 % UBM. 374.7 PPH <PROPINE>
NOZ.. 2.6 PPH <NGX-NO> C02..2.35 % NOX.. 9.5 PPM CO. 146 X NO. 7.5 PPH SMORT NUMBER

COMBUSTION EFFICIENCY, CALCULATED FROM EXMAUST CHEMISTRY: 95.0219 FUEL: RIR RATIO, CALCULATED FROM EXMAUST CHEMISTRY: .012496 REMARKS:

Figure 59 - Test Report of Experiment No. 52

U. S. ARMY FUELS & LUGHICALIS PESENRCH LABORATORY 6 1 TURBINE CONGUSTOR FACILITY 1	STUDY OF FUEL EHULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATES ** 1 U.S. NAVY AIR ENGINEERING CENTER	TIME: 1:31 COMBUSTOR STSTEM: T-63 POWER POINT: 10x TFST FUEL: JPSTHOR EMULSIFIED WITH 15% MATER	##### EXPERIMENTAL TEST CONDITIONS +++++ NVERAGE \$10.DEV, BESIRED LAST SCAN NVERAGE \$10.DEV, BEST SCAN 13.3 .88 390.00 319.69 AIR FLOW RATE, LBS/SEC 1.346 .888 1.326 1.336 FUEL FLOW RATE, LBS/SEC 1.969 .0929 .993 AIR FLOW LOADING FACTOR 1.1508 .0136 1.1508 1.1479		** OUTPUT TEMPERATURE SURVEY ** ** OUT AVC= 978.3 DEC F ** SOT AVC= 9
U. S. ARNY FUELS & LUBRICHITO RESEARCH LABORATORY TURBINE COMBUSTOR FACILITY	*** STUDY OF FUEL EMULSIONS FOR THE REDUCTION OF EXHAUST PARTICULATES ** U.S. MAYY AIR ENGINEERING CENTER	DH-Z: 6/28/76 CS-80/SYOR SYSTEM: T-63 POMER POINT: 18% TECT FUEL: JPS-HBR	#### EXPERIMENTAL IES! COMDITIONS ***** BESIRED LAST SCAN INLES AIR PRESSURE, PSIN 33.46 .18 31.59 32.10 INLES AIR FLON RAIE, BES/REC 1.376 .011 1.320 1.360 FUEL FLON RAIE, LBS/RIN .0155 .009 .057 .067 FUEL/AIR RAIIO .0155 .01095 .01095 .01095 AIR FLOW LOADING FACTOR I.1431 .0120 1.1406	FUEL !EMPERATURE=138.2 DEG F FUEL !EMPERATURE=138.2 DEG F + BURNER OUTLET TEMPERATURE BURNEY •	3£ [£

Figure 60 - Test Report of Experiment No. 53

Figure 61 - Test Report of Experiment No. 54

U. S. ARMY FUELS & LUBRICANTS RESERVEN LABORATORY TURBING COMBUSTOR FACILITY

AND STUDY OF FUEL ENULSIONS FOR THE REDUCTION OF EXHAUST PRETICULATES ... U.S. NAVY BIR ENGINEERING CENTER

DATE: 6.28/7: CORBASTOP SYSTEM: T-63 POMER POINT: 10% TEST FUEL: JPS-HOR EMALSIFIED HITH 30% WATER FUEL PRESSURE= 170 FUEL TEMPERATURE=150.9 DEG F * BURNER OUTLET TEMPERATURE SURVEY *

* EXHAUST CHEMISTRY *

CO. . 160 % CO2..1.95 % O2..18.1 % UBH. \$11.3 PPH < PRUPRIE NG: 4.4 PPH NOX.. 6.6 PPH NO2.. 2.2 PPH <NOX-NO) SHOSE HENGER: 1.8

Figure 62 - Test Report of Experiment No. 55

Exp. #	ΔΡ	<u>sn</u>
0 1	(neat fuel)	24.4
2 2	2600	16.9
□ 2 ♦ 3	1600	17.0
Δ 4	200	17.3

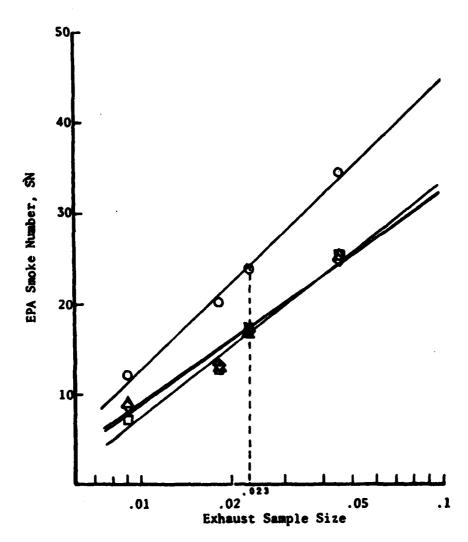


Figure 63 - Smoke Number Evaluations for Experiments 1, 2, 3, and 4 Showing the Effect of Homogenizer Pressure Drop on Smoke Reduction

E	хр. /	Surfactant Concentration	<u>\$.v</u>
\	2	2.0	16.9
0	5	0.0	25.5
Δ	6	1.0	18.5
	7	0.5	21.5
∇	8	0.0	27.1

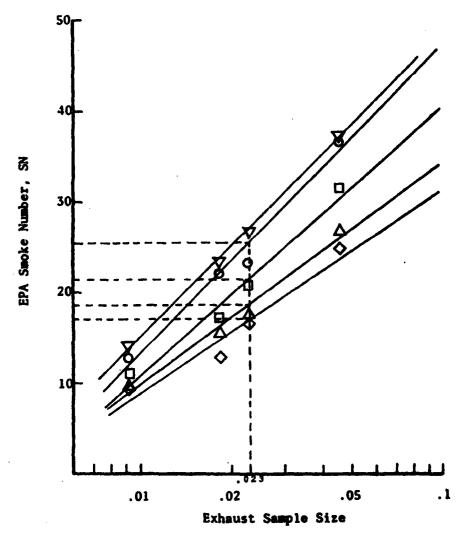


Figure 64 - Smoke Number Evaluations for Experiments 2, 5, 6, 7, and 8 Showing the Effect of Surfactant Concentration for an Emulsion of 10% Water

E	xt. #	Surfactant Concentration	SN
0	9	0	27.6
	10	2	26.6

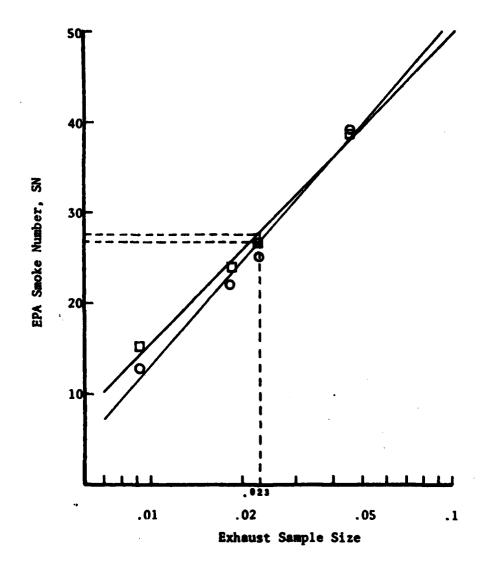


Figure 65 - Smoke Number Evaluations for Experiments 9 and 10 Showing the Effect of Just the Surfactant on Smoke Level

E	хр.	Surfactant Concentration	SN
0	11	0.0	27.2
Δ	13	2.0	21.4
	14	1.0	23.0
\Diamond	15	0.5	24.0

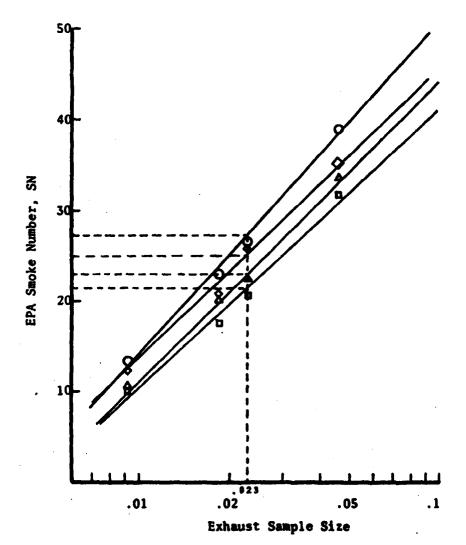


Figure 66 - Smoke Number Evaluations for Experiments 11, 13, 14, and 15 Showing the Effect of Surfactant Concentration for an Emulsion of 5% Water

Exp.#		Fuel System	SN
•	16	JP5-HA	33.6
ō	17	JP5-HA emulsified w/10% water	23.8
Ě	18	JP5-HBR	25.0
ō	19	JP5-HBR emulsified w/10% water	15.4

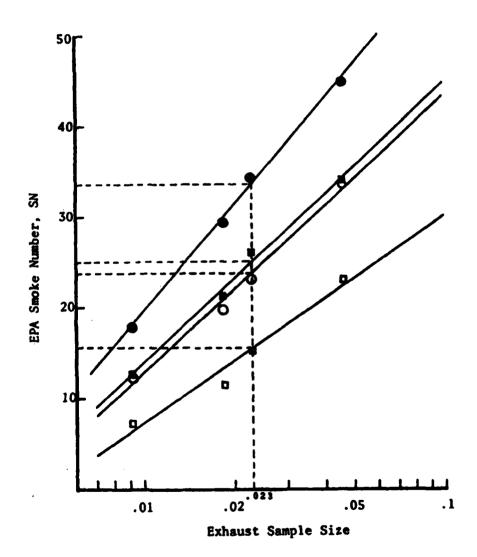


Figure 67 - Smoke Number Evaluations for Experiments 16, 17, 18, and 19 Showing Sensitivity of Concept to Fuel Type

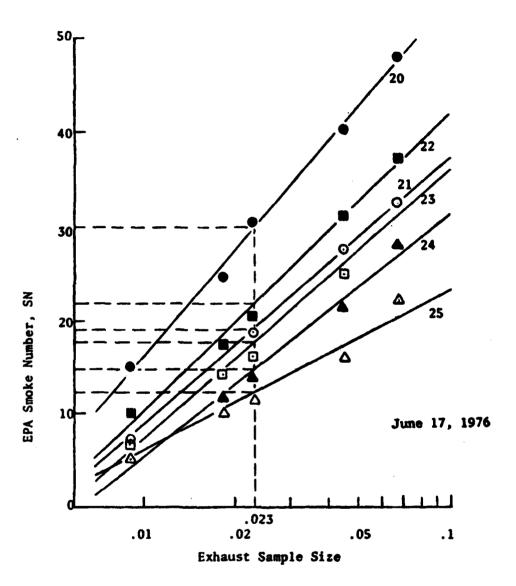


Figure 68 - Smoke Number Evaluations for Experiments 20, 21, 22, 23, 24, and 25

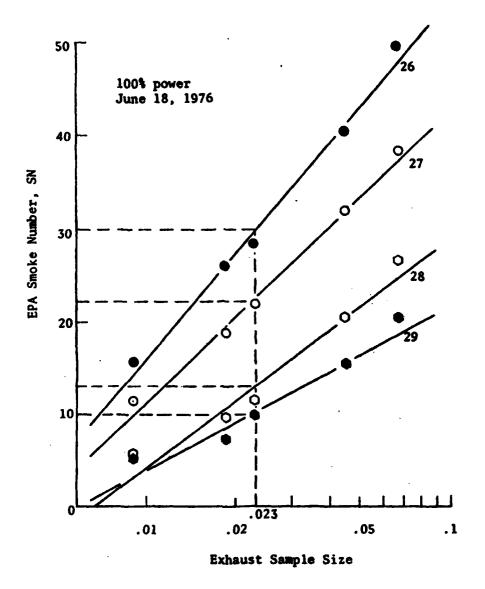


Figure 69 - Smoke Number Evaluations for Experiments 26, 27, 28, and 29

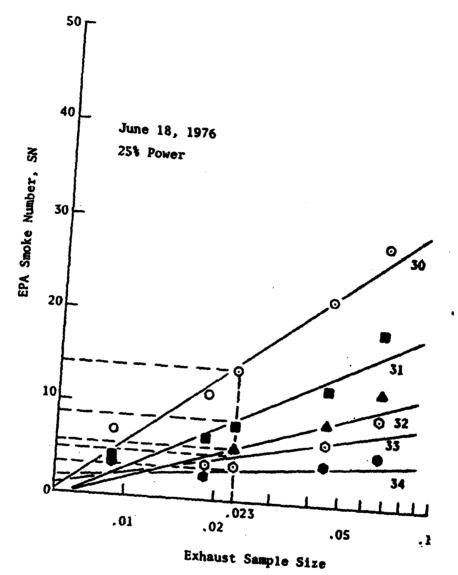


Figure 70 - Smoke Number Evaluations for Experiments 30, 31, 32, 33, and 34

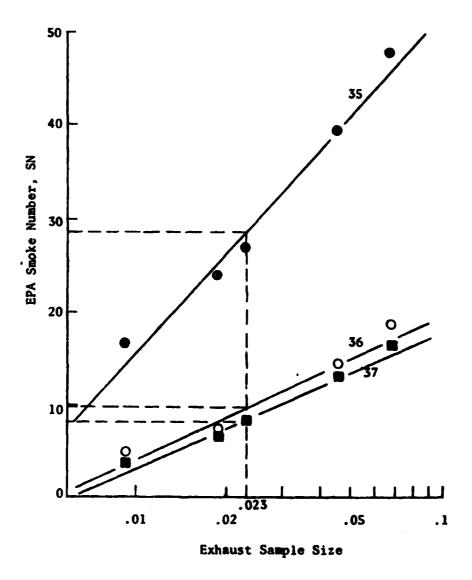


Figure 71 - Smoke Number Evaluations for Experiments 35, 36, and 37

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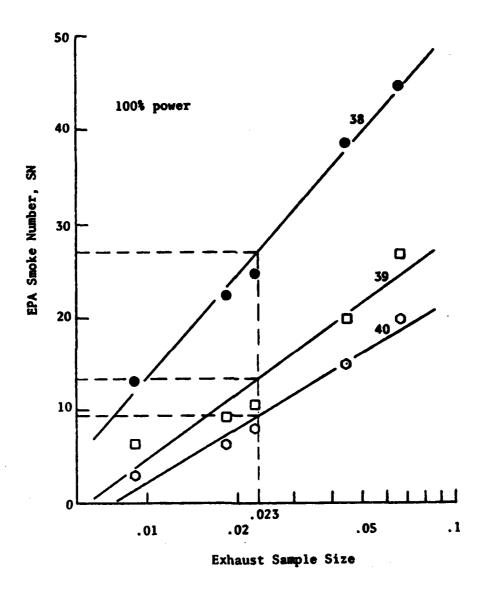


Figure 72 - Smoke Number Evaluations for Experiments 38, 39, and 40

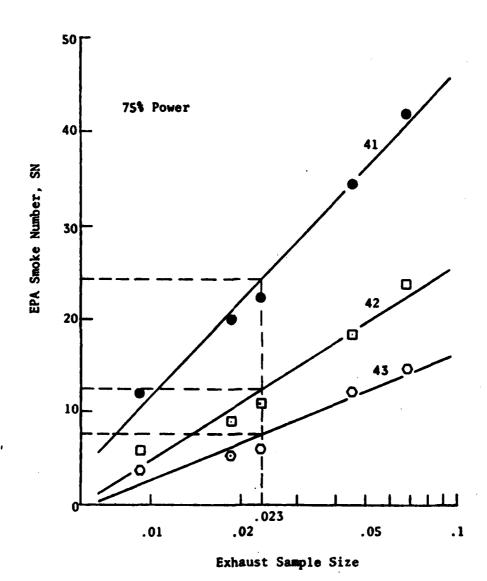


Figure 73 - Smoke Number Evaluations for Experiments 41, 42, and 43

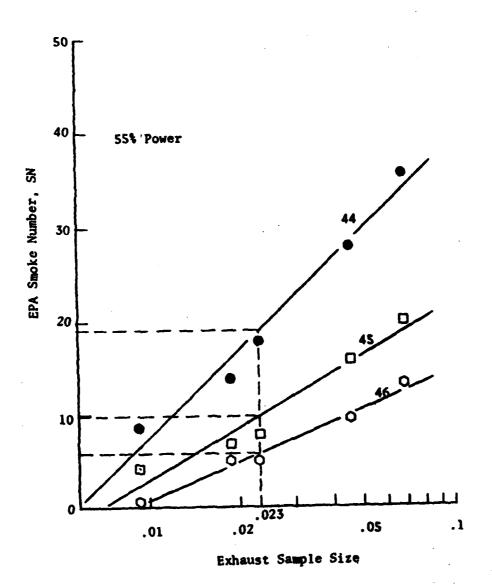


Figure 74 - Smoke Number Evaluations for Experiments 44, 45, and 46

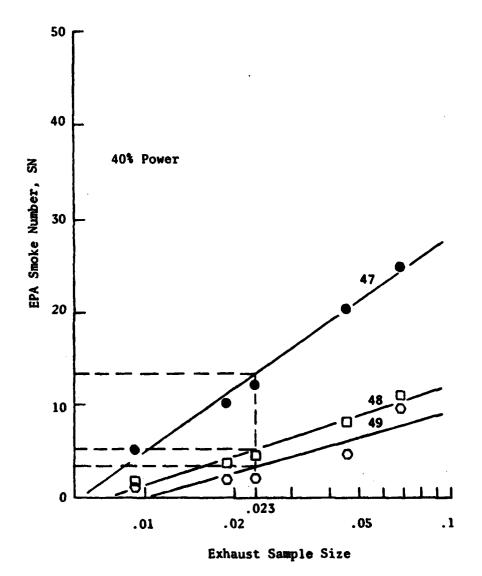


Figure 75 - Smoke Number Evaluations for Experiments 47, 48, and 49

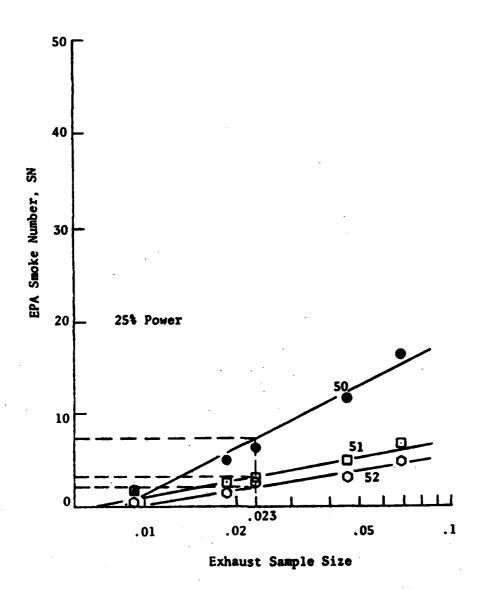


Figure 76 - Smoke Number Evaluations for Experiments 50, 51, and 52

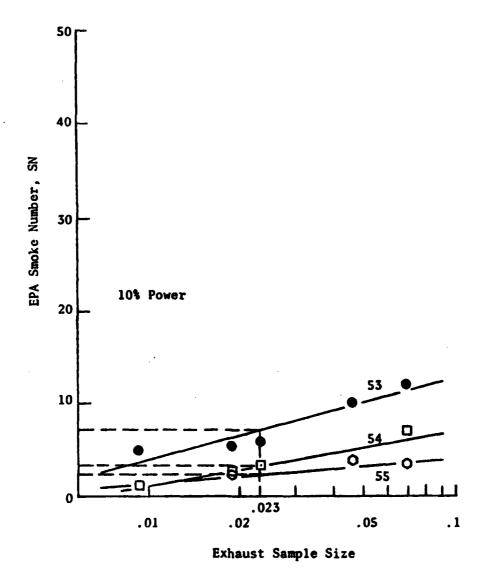


Figure 77 - Smoke Number Evaluations for Experiments 53, 54, and 55

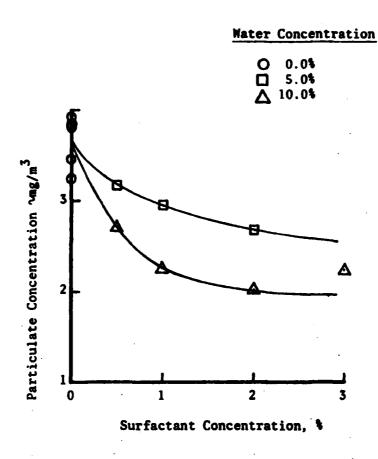


Figure 78 - Summary of Effects of Emulsion Characteristics on the Reduction of Exhaust Particulates, Program Phases 1-3

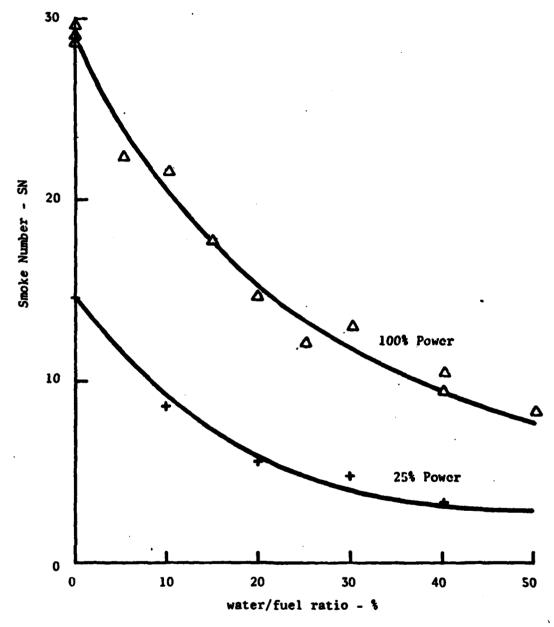
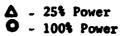


Figure 79 - Summary of Effects of Water Concentration on the Reduction of Smoke Number, Program Phases 4-7



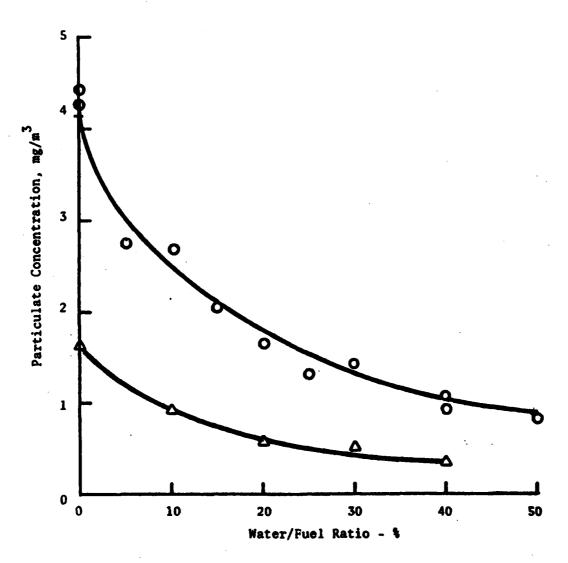


Figure 80 - Summary of Effects of Water Concentration on Particulate Concentration

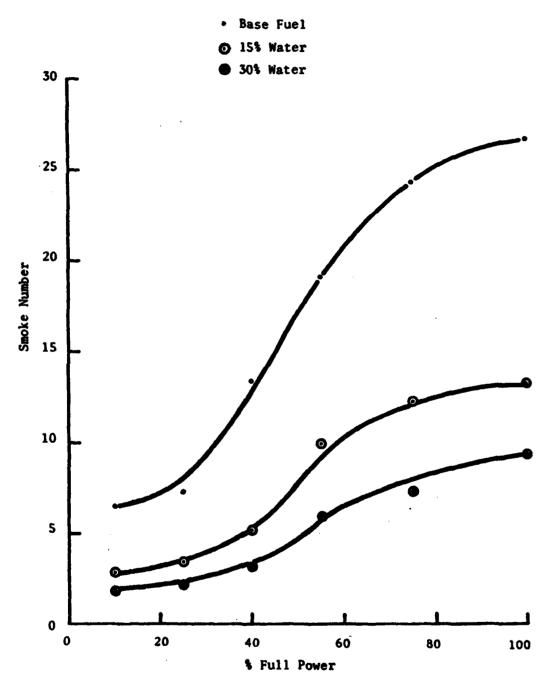


Figure 81 - Summary of Effects of Fuel/Water Emulsions on Smoke Number Throughout Engine Power Spectrum

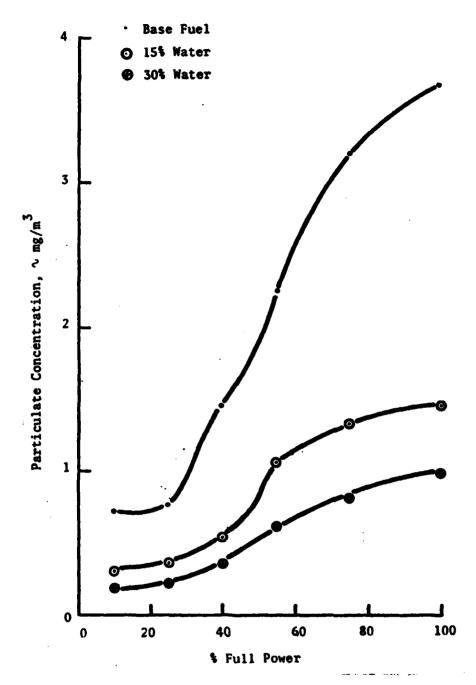


Figure 82 - Summary of Effects of Emulsion Characteristics on Particulate Concentration Over Engine Power Spectrum

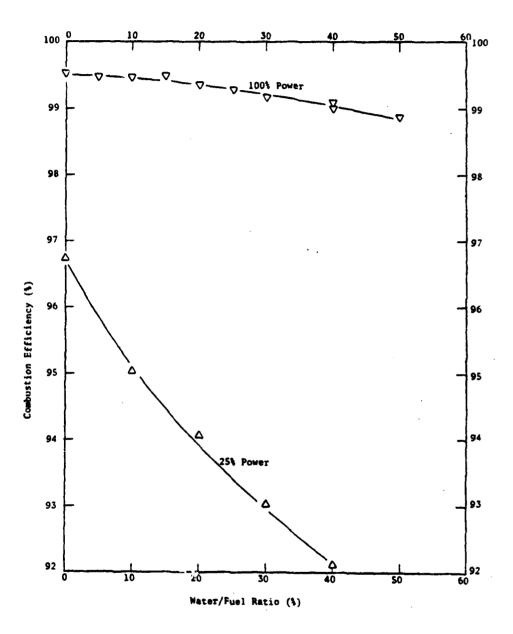


Figure 83 - Effect of Water Concentration on Combustion Efficiency

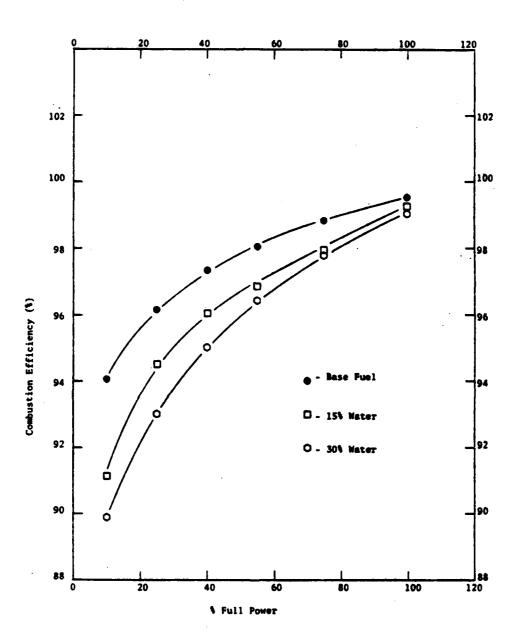


Figure 84 - Effect of Water Concentration on Combustion Efficiency Over the Engine Power Spectrum

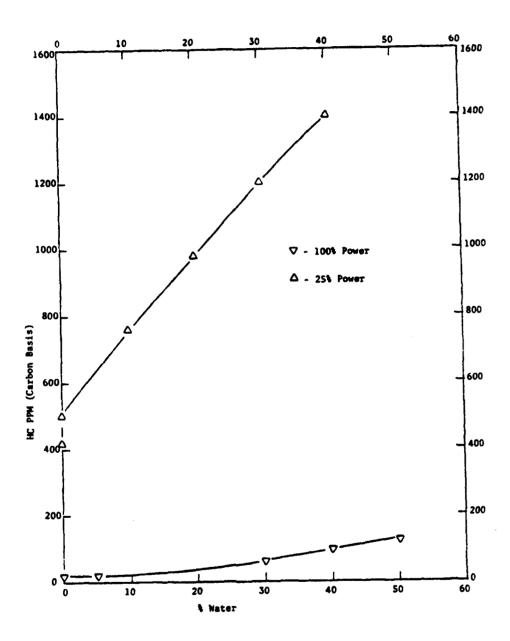


Figure 85 - Effect of Water Concentration on Hydrocarbon Emissions

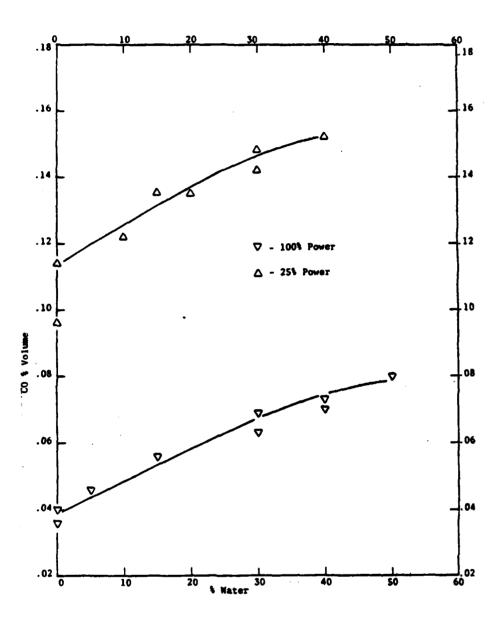


Figure 86 - Effect of Water Concentration on Carbon Monoxide Emissions

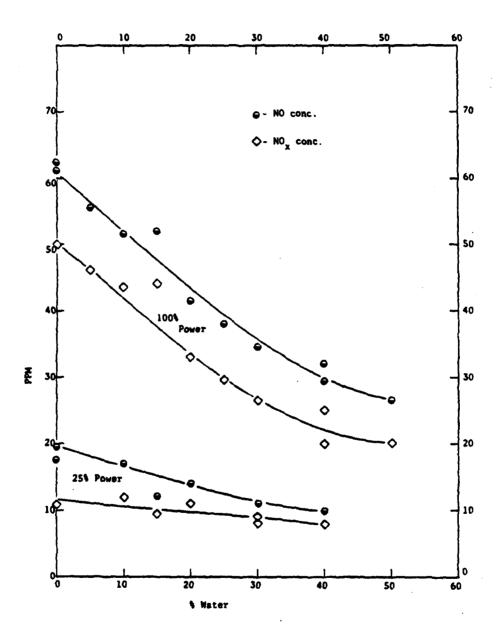


Figure 87 - Effect of Water Concentration on Oxides of Nitrogen Emissions

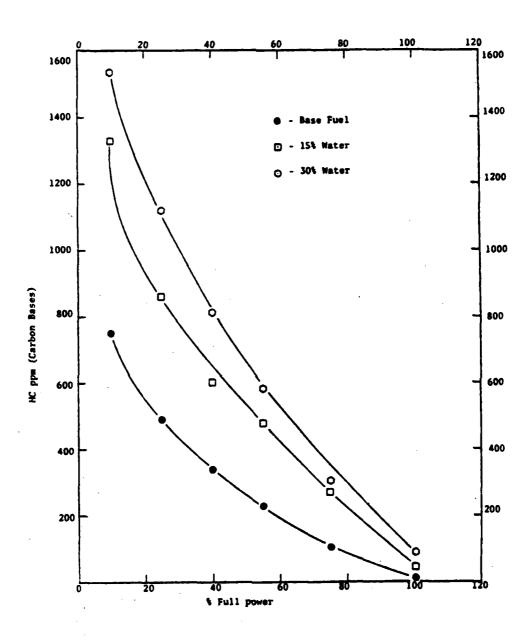


Figure 88 - Effect of Water Concentration on Hydrocarbon Emissions for Engine Power Spectrum

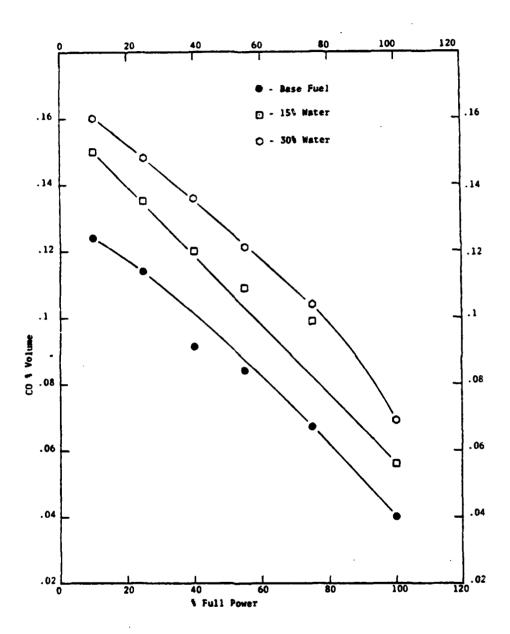


Figure 89 - Effect of Water Concentration on Carbon Monoxide Emissions for Engine Power Spectrum

The street with the same

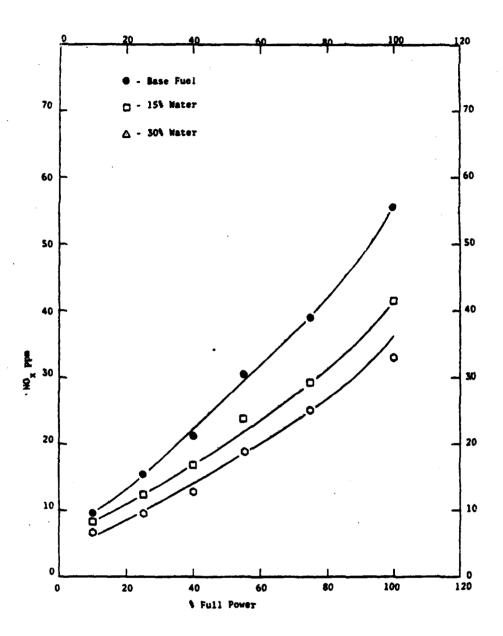


Figure 90 - Effect of Water Concentration on Oxides of Nitrogen Emissions for Engine Power Spectrum

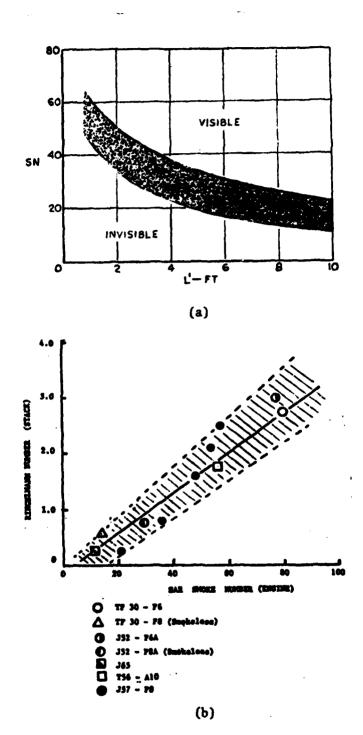


Figure 91 - Correlations of Smoke Number to Plume Visibility According to (a) Champagne (1971), (b) Kelly (1973)

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REVISION LIST

REVISION	PAGES AFFECTED	DATE OF REVISION

4MD-MAEC 5213/3B (10-77)

